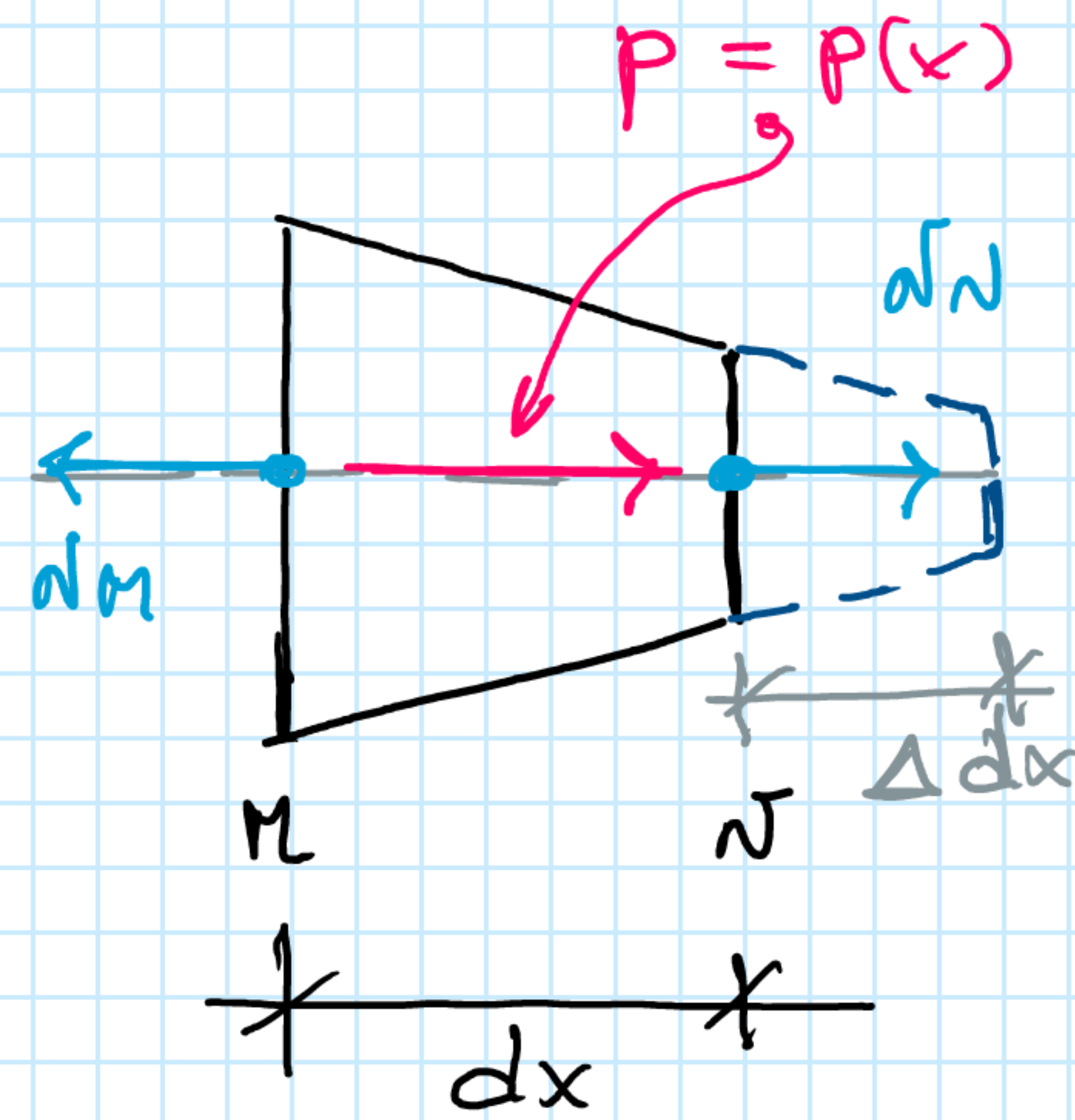
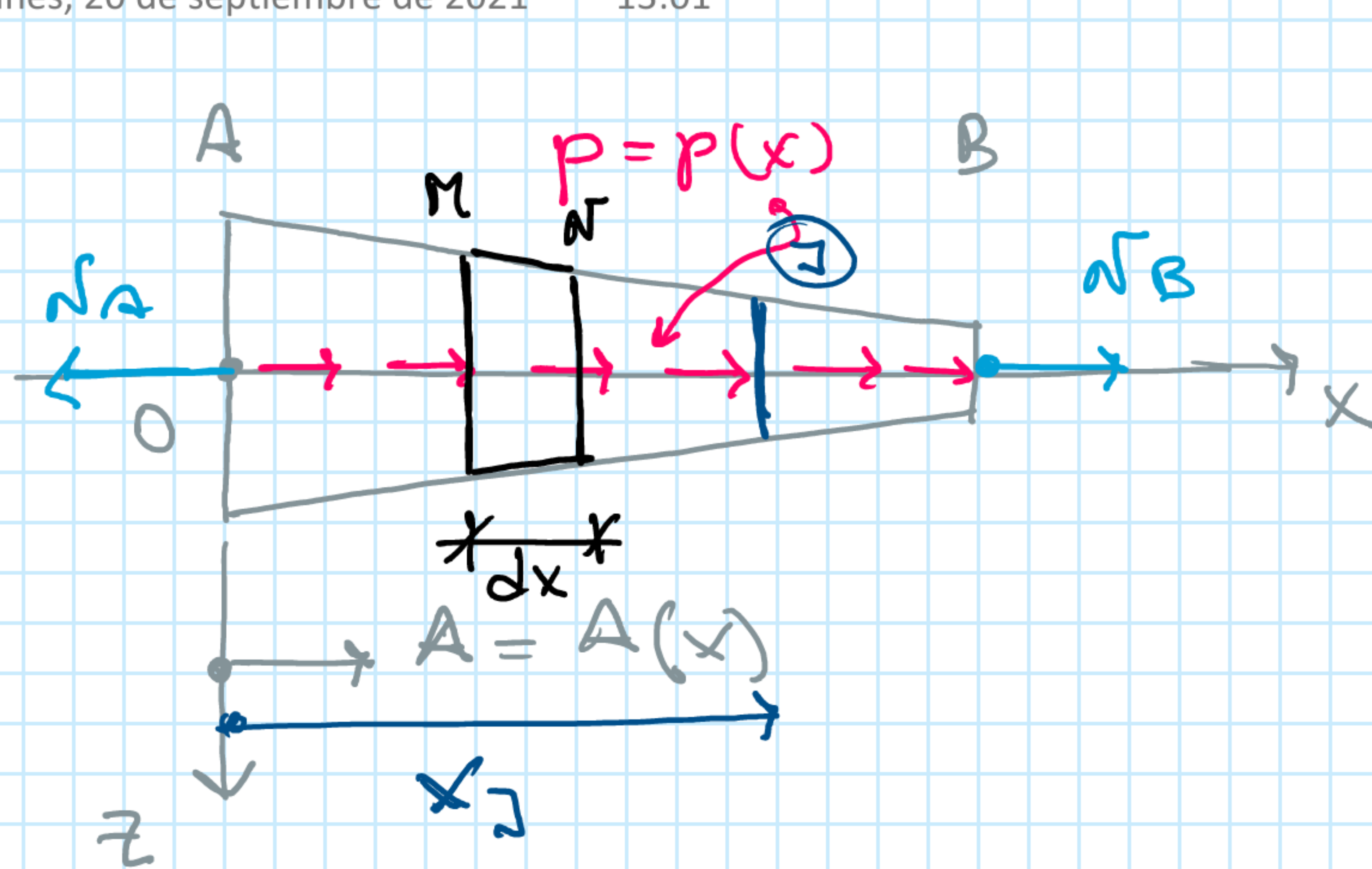


01.09 - DESPLAZAMIENTOS Y DEFORMACIONES:

lunes, 20 de septiembre de 2021 13:01

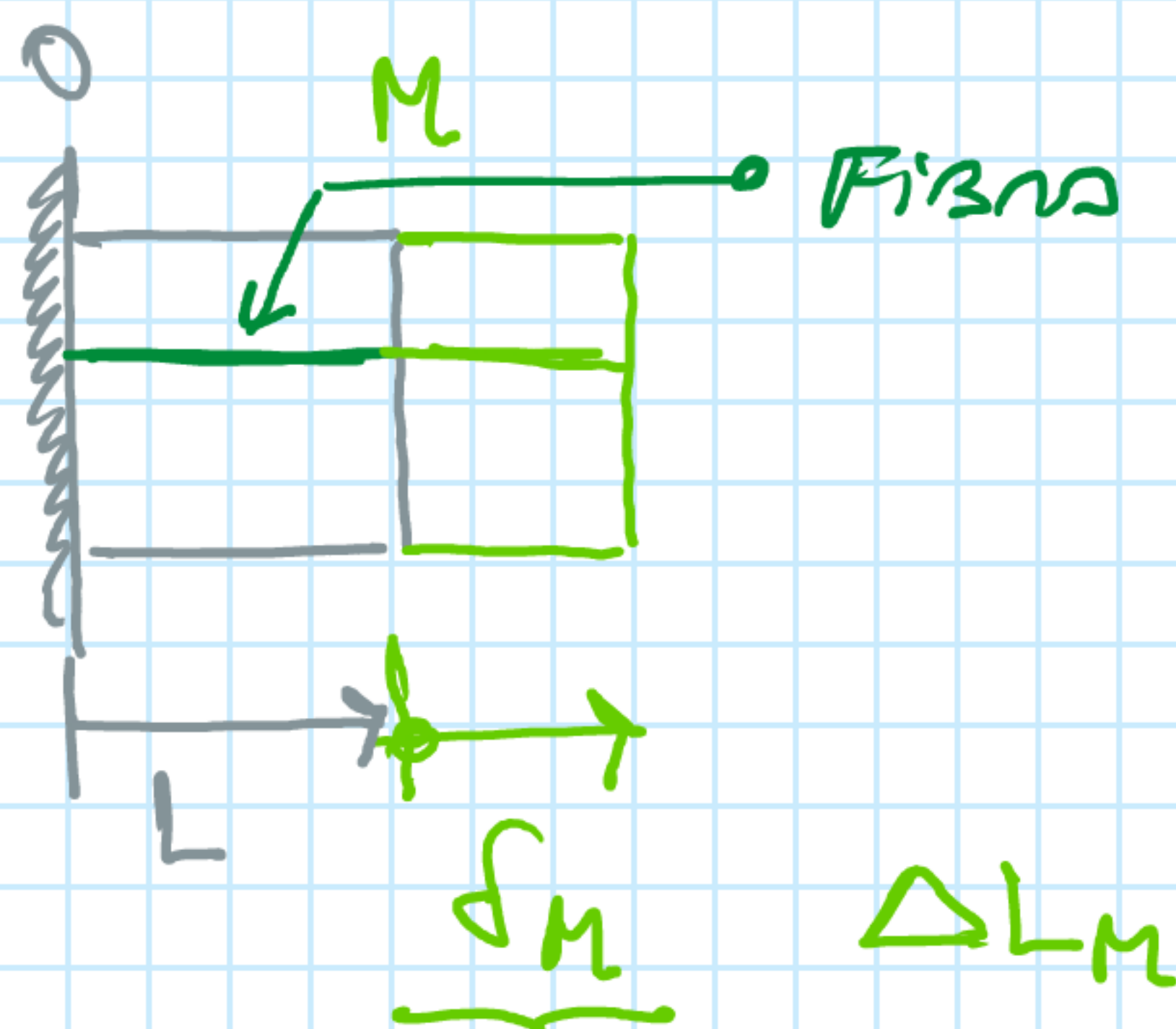


$$\frac{\Delta dx(x)}{dx} = \epsilon_{xx}(x) ; \quad \sigma_x(x) = E \epsilon_{xx}(x) \rightarrow \epsilon_{xx}(x) = \frac{\sigma_x(x)}{E}$$

$$\Delta dx(x) = \epsilon_{xx}(x) \cdot dx = \frac{\sigma_x(x)}{E} \cdot dx ; \quad \sigma_x(x) = \frac{N(x)}{A(x)}$$

$$\Delta dx(x) = \frac{N(x)}{E \cdot A(x)} \cdot dx$$

$$\Delta L(x) = \delta(x) = \int \Delta dx(x) = \int \frac{N(x)}{E A(x)} dx$$



δ : DESPLAZAMIENTOS DE LOS PUNTOS

Fibra $\overline{OM}$ = LONGITUD INICIAL = $L = l_0$
 " final = $L + \Delta L = l_f$

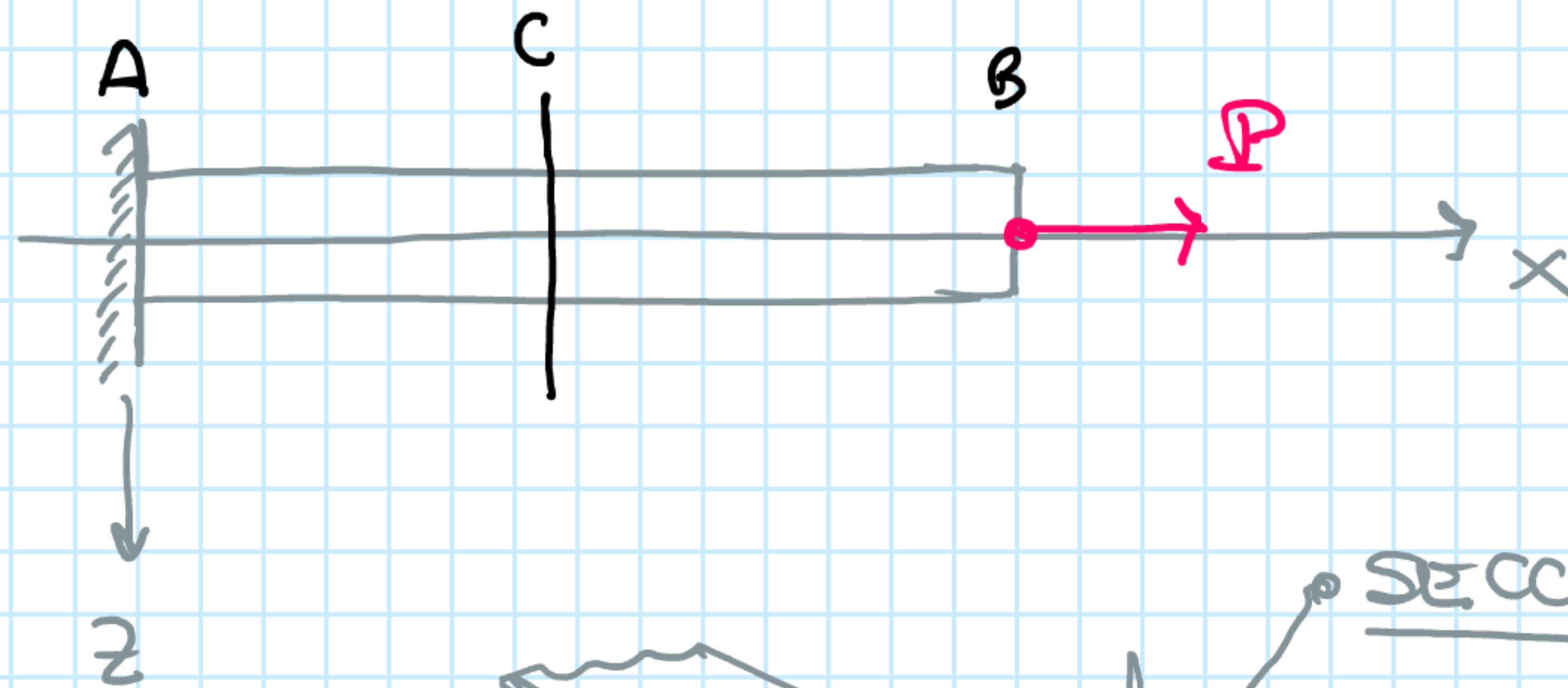
$$\epsilon_{xx} = \frac{d\delta(x)}{dx} = \frac{d[\Delta L(x)]}{dx} = \frac{N(x)}{E \cdot A(x)}$$

↑
 válido si la cosa deformado
 es la cosa fuerza.

01.10 - VOLUMEN DE TENSIONES:

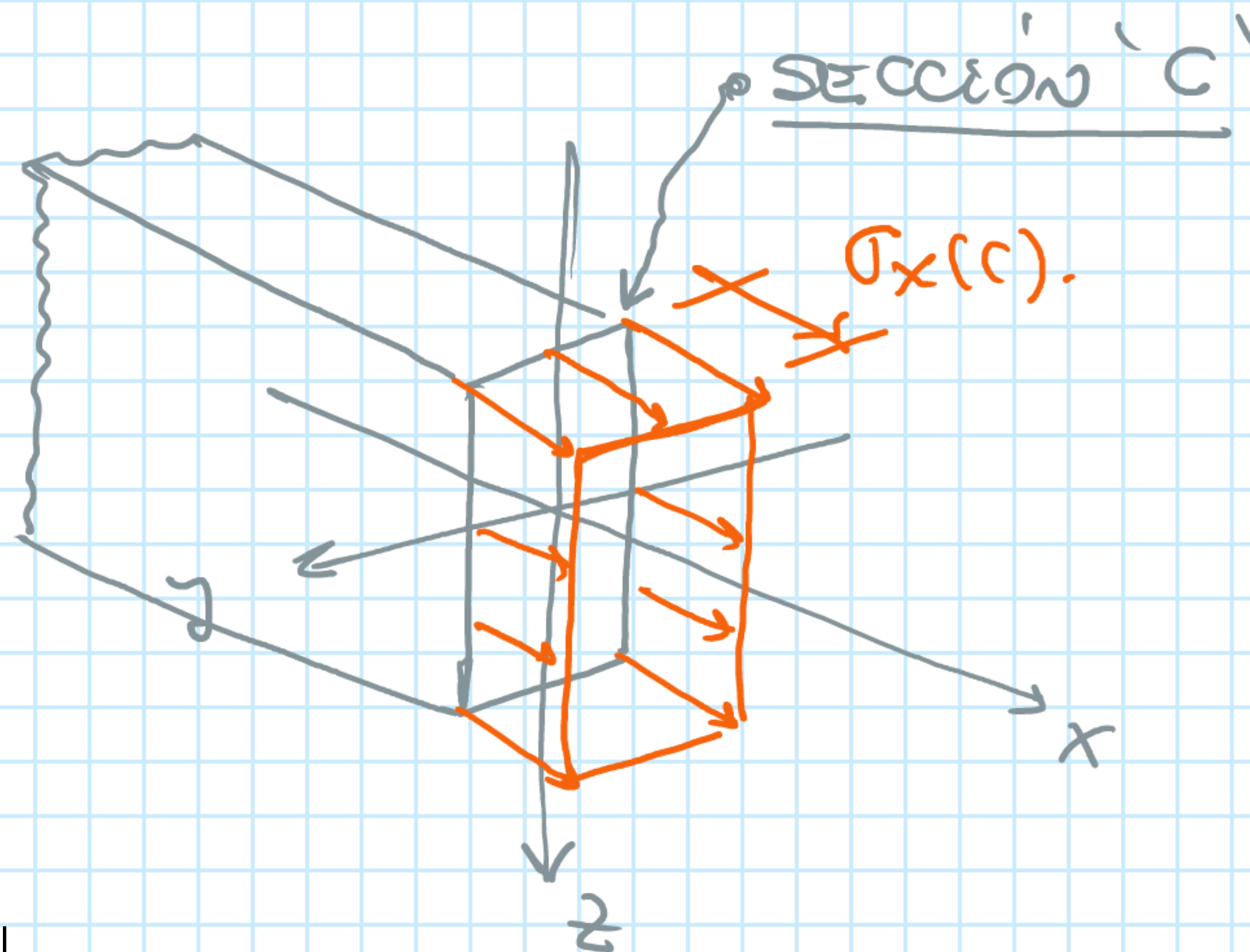
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$$\sigma_c = P.$$

$$\sigma_x(c) = \frac{\sigma_c}{A_c}$$



$$\sigma_c = \int_A \sigma_x dA$$

01.11 - RESUMEN DE EXPRESIONES:

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1) MAGNITUDES ESTÁTICAS:

$$N = N(x)$$

$$\sigma_x = \frac{N(x)}{A(x)}$$

2) MAGNITUDES CINEMÁTICAS

$$\delta(x) = \Delta L(x) = \int \frac{\sigma(x)}{E A(x)} dx. \quad (I)$$

$$\epsilon_{xx}(x) = \frac{d\delta(x)}{dx} = \frac{d\Delta L(x)}{dx}.$$

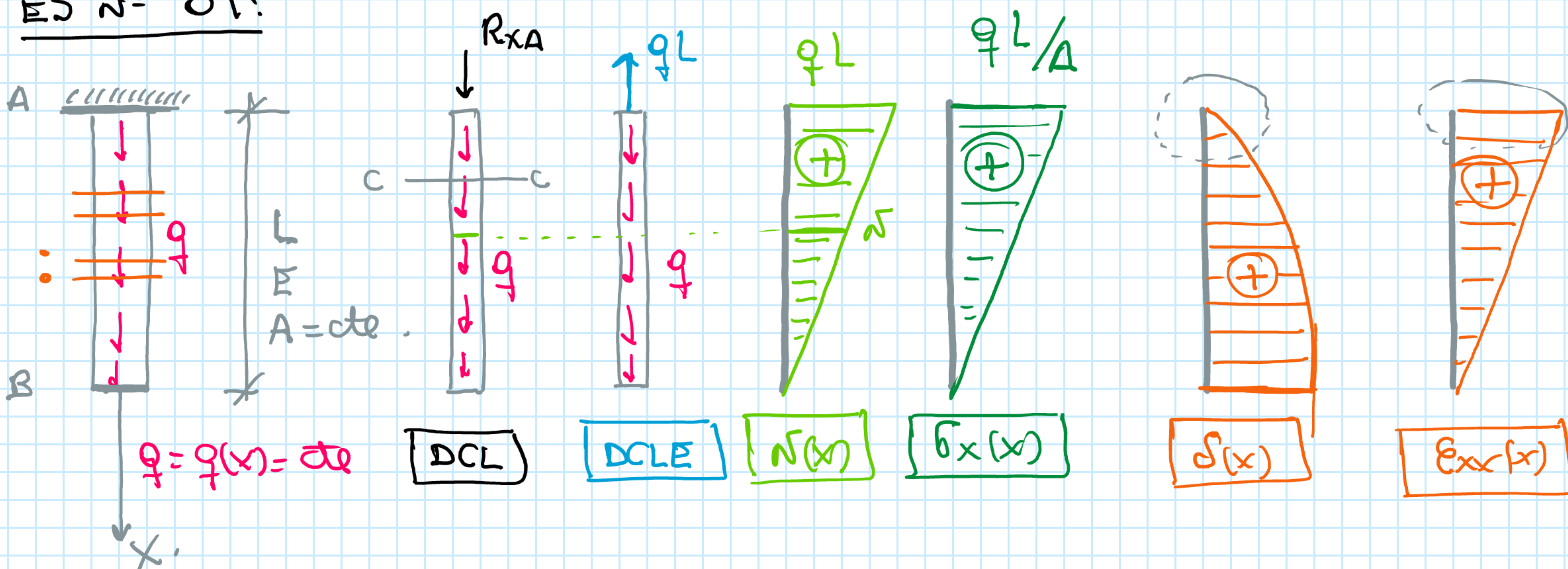
$$\epsilon_{xx}(x) = \frac{\sigma(x)}{E A(x)} \quad (II)$$

(I), (II) son válidas P/ la causa fuerza.

01.12 - EJERCICIOS:

lunes, 20 de septiembre de 2021 13:31

EJ N° 01:



RVE $\sum F_x = 0$ $q \cdot L + R_{xA} = 0 \rightarrow R_{xA} = -qL$ $\uparrow$

Free-body diagram of a section of length x from the free end B:

$\sum F_x = 0$ $N(x) + q \cdot x - qL = 0$

$N(x) = qL - qx \rightarrow N(x) = q(L-x)$

Boundary conditions:

$$\begin{cases} N(x=0) = qL \\ N(x=L) = 0 \end{cases}$$

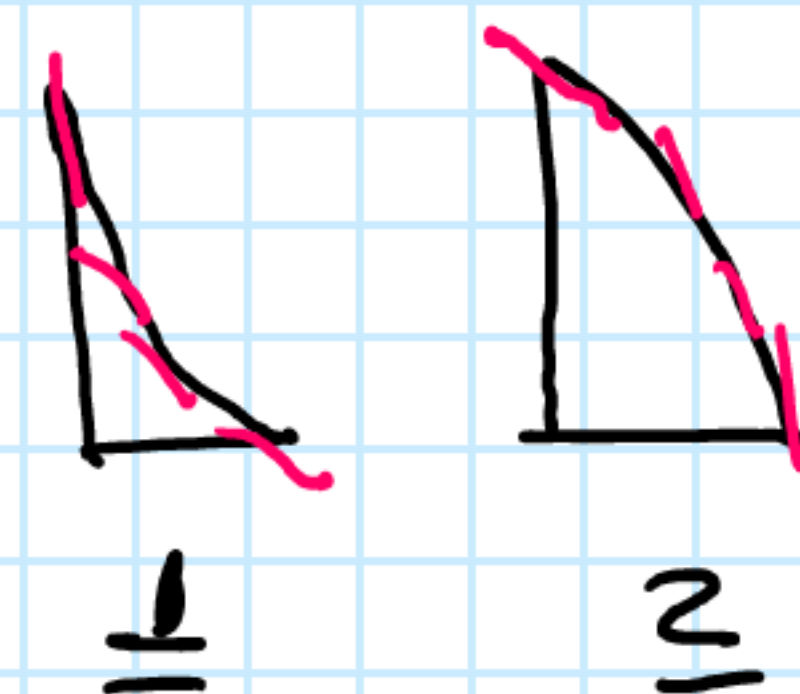
TENSIONES:

$$\sigma_x(x) = \frac{N(x)}{A(x)} = \frac{N(x)}{A} = \frac{q(L-x)}{A}$$

DESPLAZAMIENTOS:

$$\delta(x) = \Delta L(x) = \int \frac{N(x)}{EA(x)} dx = \frac{1}{EA} \int q(L-x) dx = \frac{q}{EA} \int (L-x) dx$$

$$\delta(x) = \Delta L(x) = \frac{q}{EA} \left[Lx - \frac{x^2}{2} \right]$$



$$\delta(x=0) = 0$$

$$\delta(x=L) = \frac{q}{EA} \left[L^2 - \frac{L^2}{2} \right] = \left[\frac{1}{2} \frac{q}{EA} L^2 \right]$$

DEFORMACIONES:

$$\epsilon_{xx}(x) = \frac{d\delta(x)}{dx} = \frac{q}{EA} [L - x]$$

$\epsilon_{xx}(0) = \frac{qL}{EA}$
 $\epsilon_{xx}(L) = 0$

ALUMINUM:

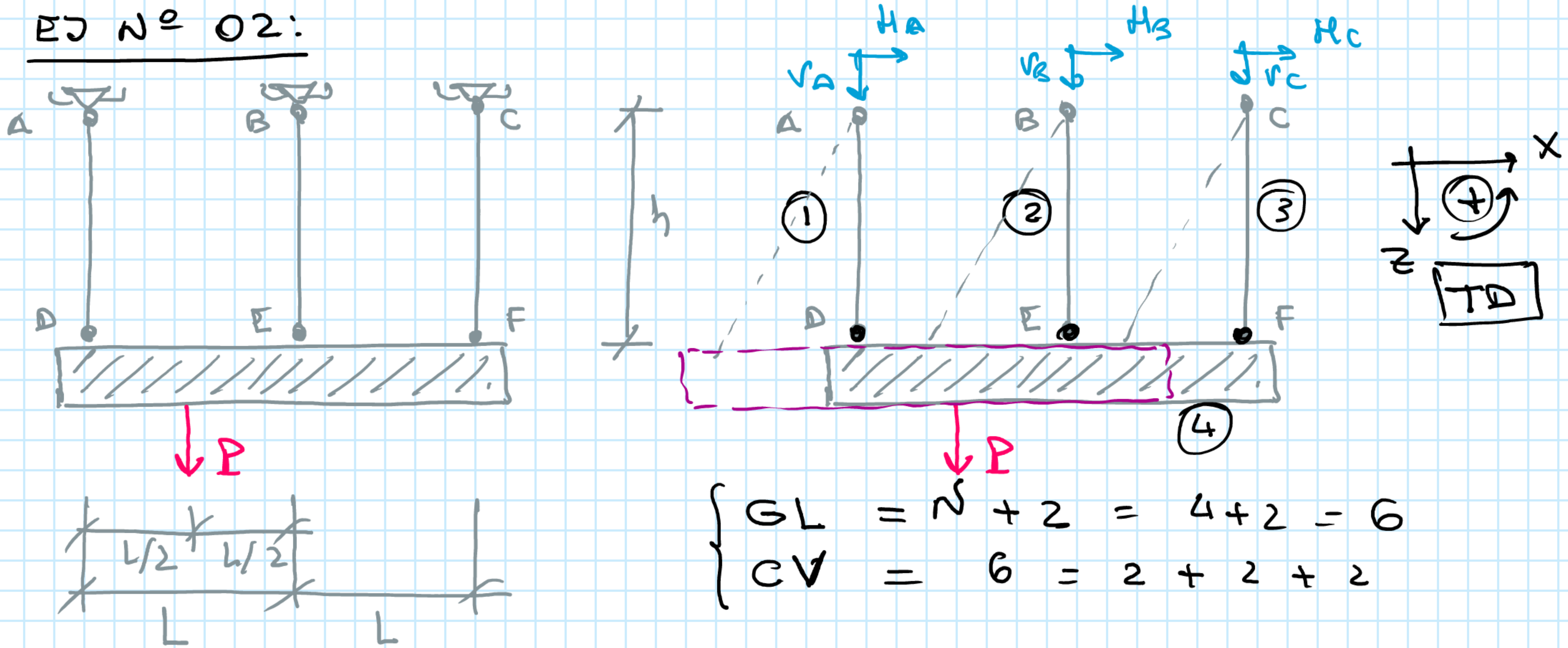
q $\downarrow$ E en aluminio lino.

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EJ N° 02:



ECS DE EQUILIBRIO RESATIVO: 'EER'

$$\sum M_{F_i, sup}^D = 0 \quad H_A \cdot h = 0 \rightarrow \boxed{H_A = 0}$$

$$\sum M_{F_i, sup}^E = 0 \quad H_B \cdot h = 0 \rightarrow \boxed{H_B = 0}$$

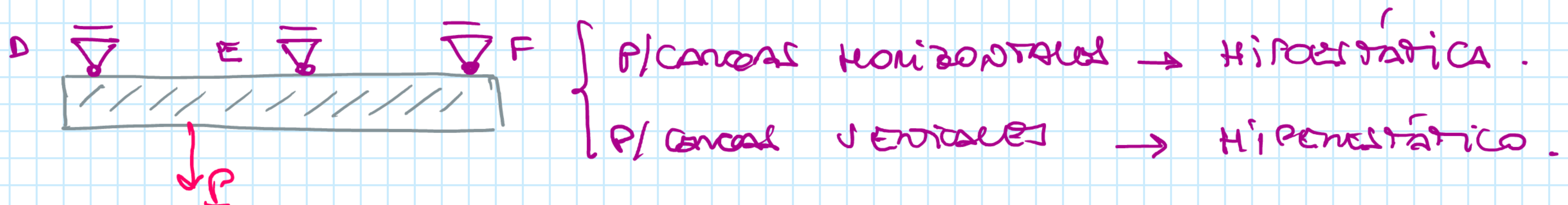
$$\sum M_{F_i, sup}^F = 0 \quad H_C \cdot h = 0 \rightarrow \boxed{H_C = 0}$$

E E A:

$$\sum F_V = 0 \quad P + V_A + V_B + V_C = 0.$$

$$\sum M_{F_i}^F = 0 \quad P \cdot \frac{3}{2}L + V_A \cdot 2L + V_B \cdot L = 0.$$

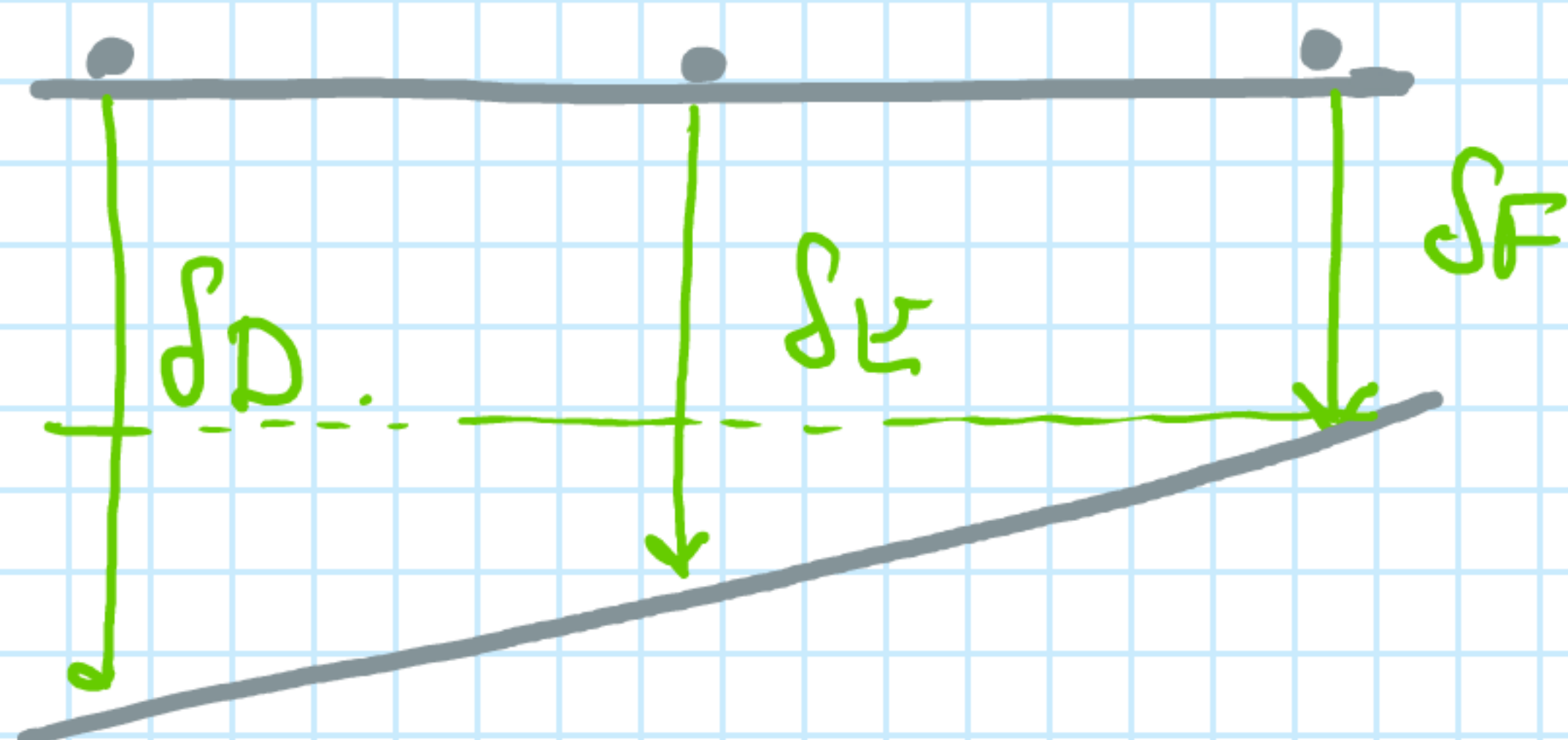
$$\sum F_H = 0 \quad H_A + H_B + H_C = 0 \rightarrow 0 = 0$$



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ECS DE COMPATIBILIDAD DE LOS DESPLAZAMIENTOS.



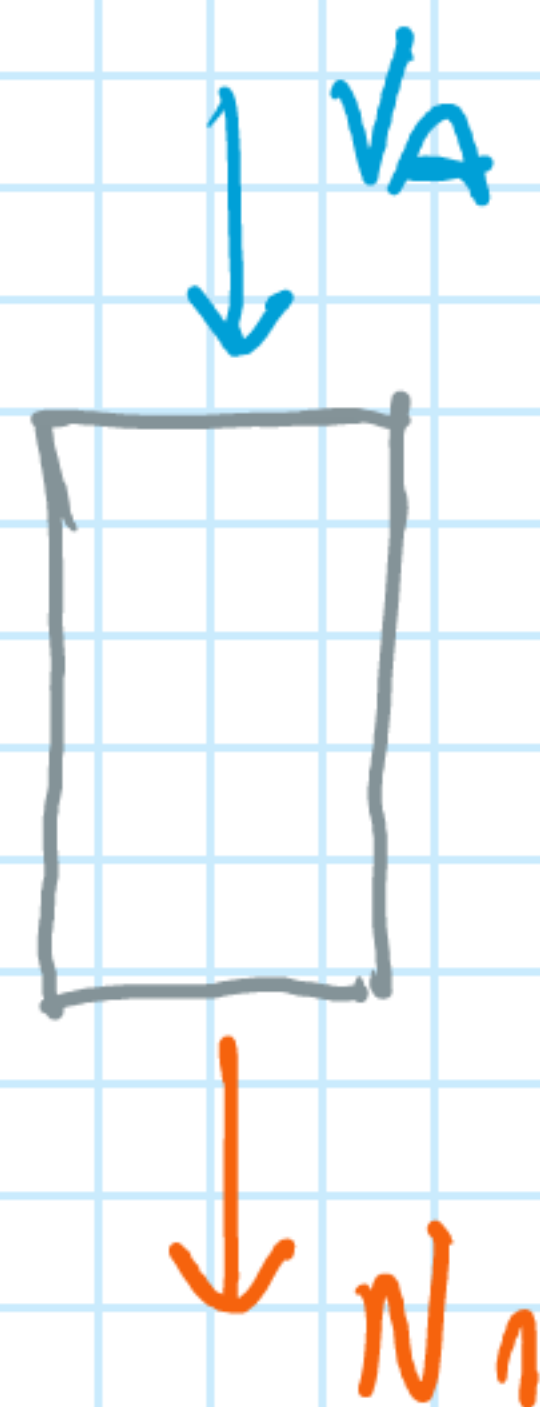
$$\frac{\delta_D - \delta_F}{2L} = \frac{\delta_E - \delta_F}{L} \leftarrow$$

$$\delta_D = \int \frac{\sigma(x)_1}{E_1 A_1} \cdot dx = \frac{\sigma_1}{E_1 A_1} \int dx = \frac{\sigma_1 h}{E_1 A_1}$$

$$\delta_E = \frac{\sigma_2 h}{E_2 A_2}$$

$$\delta_F = \frac{\sigma_3 h}{E_3 A_3}$$

Suponemos que $\begin{cases} E_1 = E_2 = E_3 = E \\ A_1 = A_2 = A_3 = A \end{cases}$



$$\sum F_x = 0 \quad V_A + \sigma_1(x) = 0 \rightarrow \boxed{\sigma_1(x) = -V_A}$$

$$\delta_D = -\frac{V_A h}{EA} \quad ; \quad \delta_E = -\frac{V_B h}{EA} \quad ; \quad \delta_F = -\frac{V_C h}{EA}$$

$$\frac{\delta_D - \delta_F}{2L} = \frac{\delta_E - \delta_F}{L} \leftarrow$$

$$\frac{-\frac{V_A h}{EA} - \left(-\frac{V_C h}{EA}\right)}{2L} = \frac{-\frac{V_B h}{EA} - \left(-\frac{V_C h}{EA}\right)}{L}$$

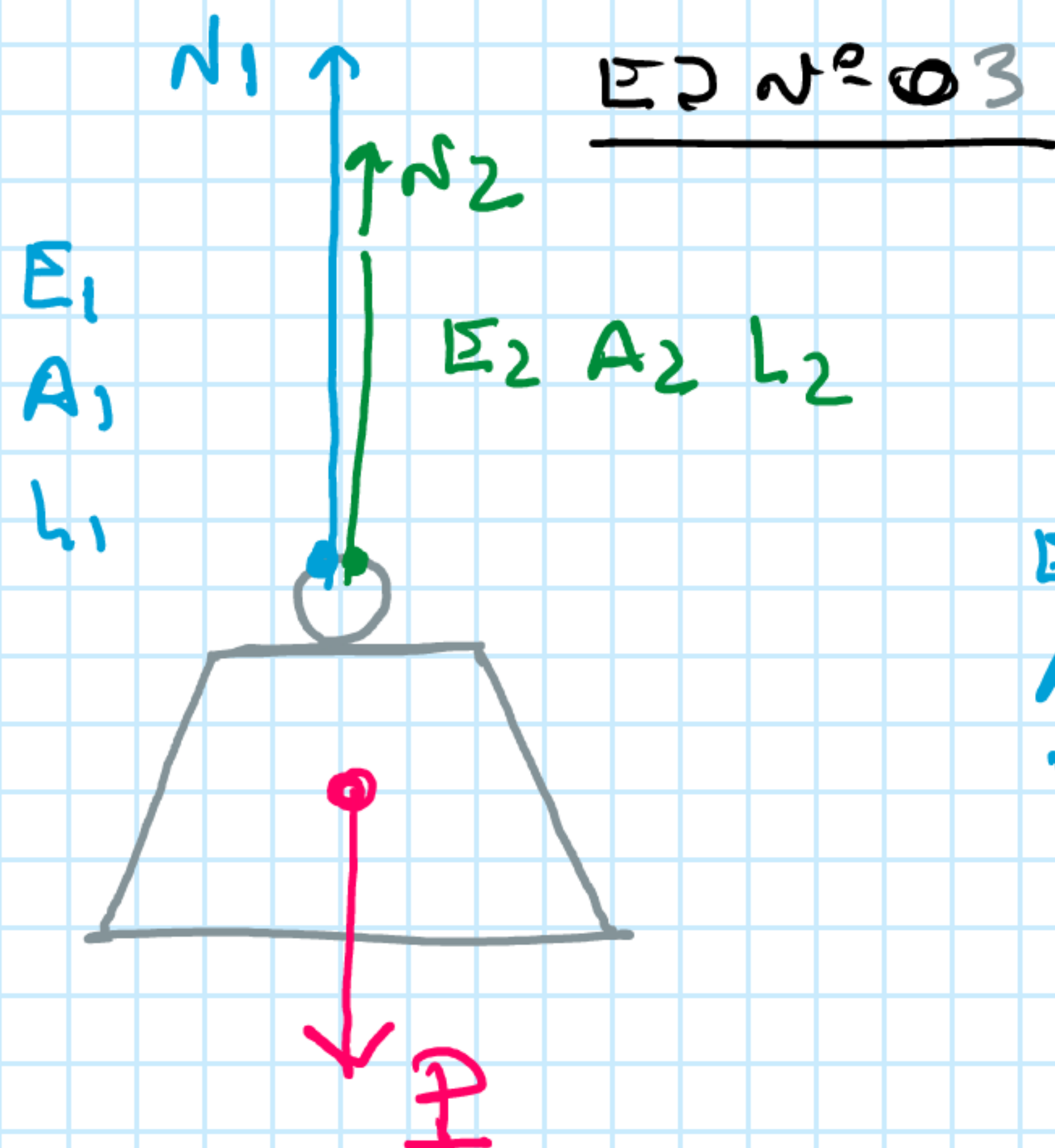
$$\frac{h}{EA} (V_C - V_A) \frac{1}{2L} = \frac{h}{EA} (V_C - V_B) \frac{1}{L}$$

$$\frac{1}{2} V_C - \frac{1}{2} V_A = V_C - V_B \rightarrow V_B - \frac{1}{2} V_A = V_C - \frac{1}{2} V_C = \frac{V_C}{2}$$

$$\boxed{V_C = 2V_B - V_A}$$

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$$\sum F_x = 0$$

$$P - n_1 - n_2 = 0.$$

$$\Delta L_1 = \Delta L_2$$

$$\frac{n_1 L_1}{E_1 A_1} = \frac{n_2 L_2}{E_2 A_2}$$

$$n_1 = \frac{n_2}{\frac{E_2 A_2}{L_2}} \cdot \left(\frac{E_1 A_1}{L_1} \right)^{K_1}$$

$$\left\{ \begin{array}{l} n_1 = n_2 \cdot \frac{K_1}{K_2} \quad K_1 > K_2 \rightarrow \\ \quad \quad \quad \rightarrow n_1 > n_2 \\ K_1 < K_2 \rightarrow n_1 < n_2. \end{array} \right.$$

$$f(x) = \Delta L(x) = \int \frac{n(x)}{EA} dx \quad \text{si } \left. \begin{array}{l} n = \text{cte.} \\ E = \text{cte.} \\ A = \text{cte.} \end{array} \right\} \quad f(x) = \frac{nL}{EA} \rightarrow$$

$$\rightarrow f(x) = \frac{n}{\frac{EA}{L}} = \frac{n}{K} \quad \text{si } \left\{ \begin{array}{l} K \uparrow \rightarrow f(x) \downarrow \\ K \downarrow \rightarrow f(x) \uparrow \end{array} \right.$$

$$N = (K) f(x) \rightarrow$$

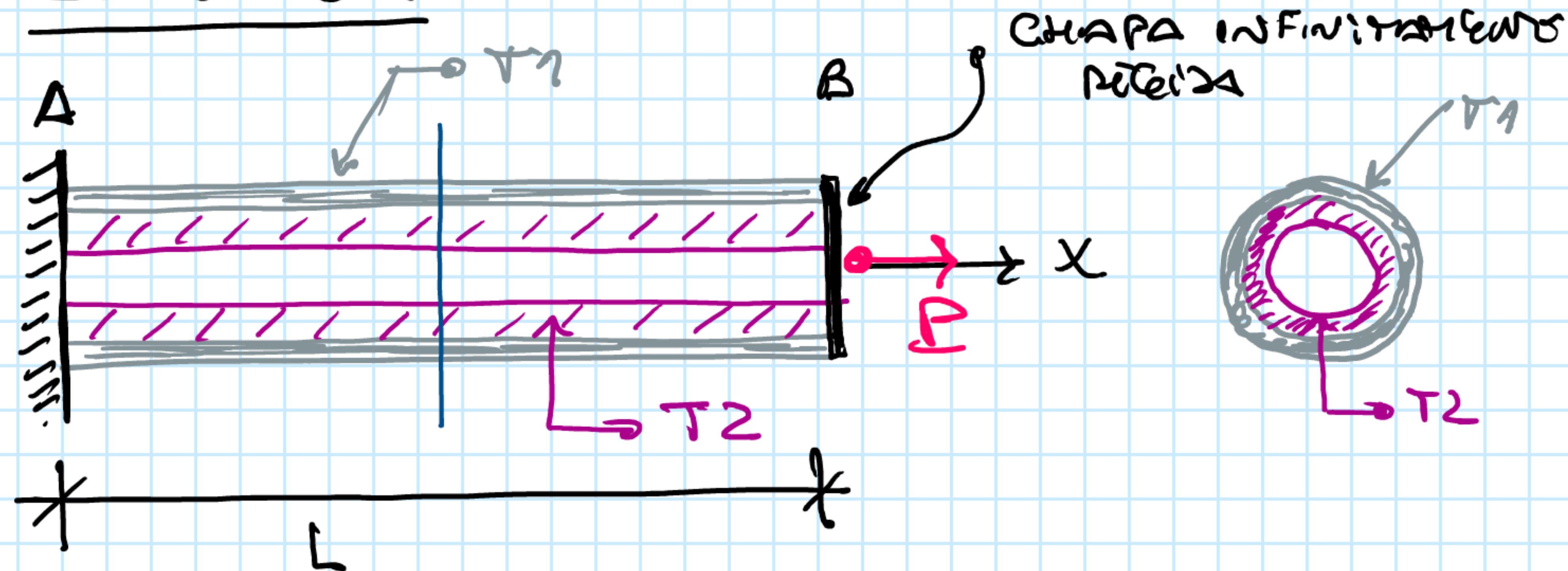
$\hookrightarrow K = \frac{EA}{L} = \text{resistencia a la succi3n axial.}$

EN ESTRUCTURAS HIPERESTÁTICAS, LOS ELEMENTOS ESTRUCTURALES
MÁS RÍGIDOS VANAN MAYORES ESFUERZOS.

01.12 -

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ES. N° 04 :



$$\sum F_x = 0 \quad R_{xA} + P = 0 \quad \rightarrow \quad \boxed{R_{xA} = -P} \quad \leftarrow$$



$$\boxed{P - N_{T1} - N_{T2} = 0}$$

HIPERESTÁTICO. POR VÍNCULO INTERNO

$$\Delta L_1 = \Delta L_2.$$

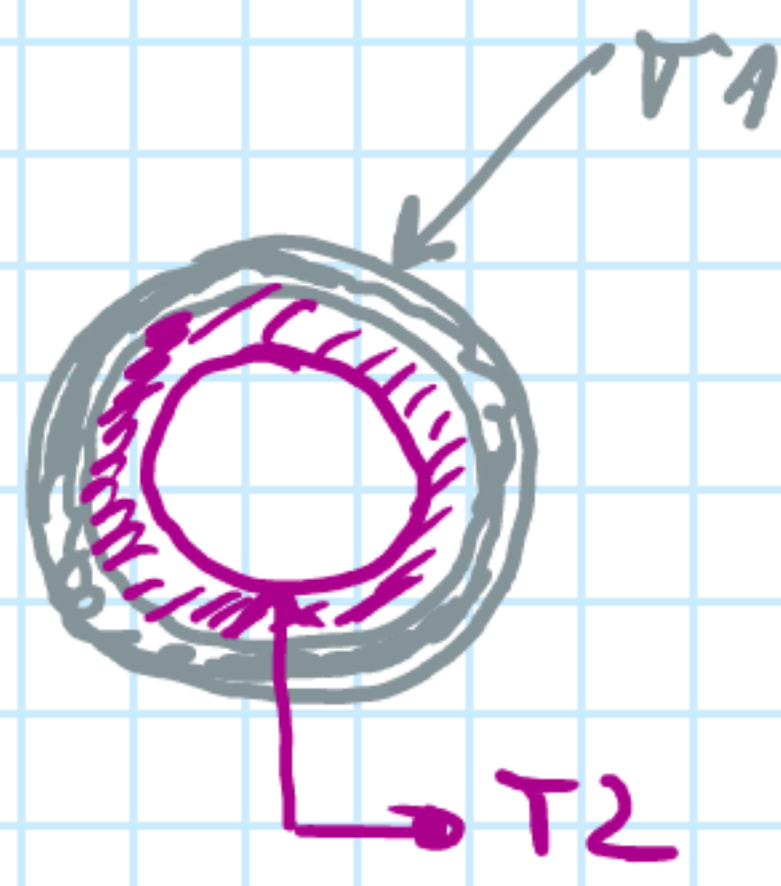
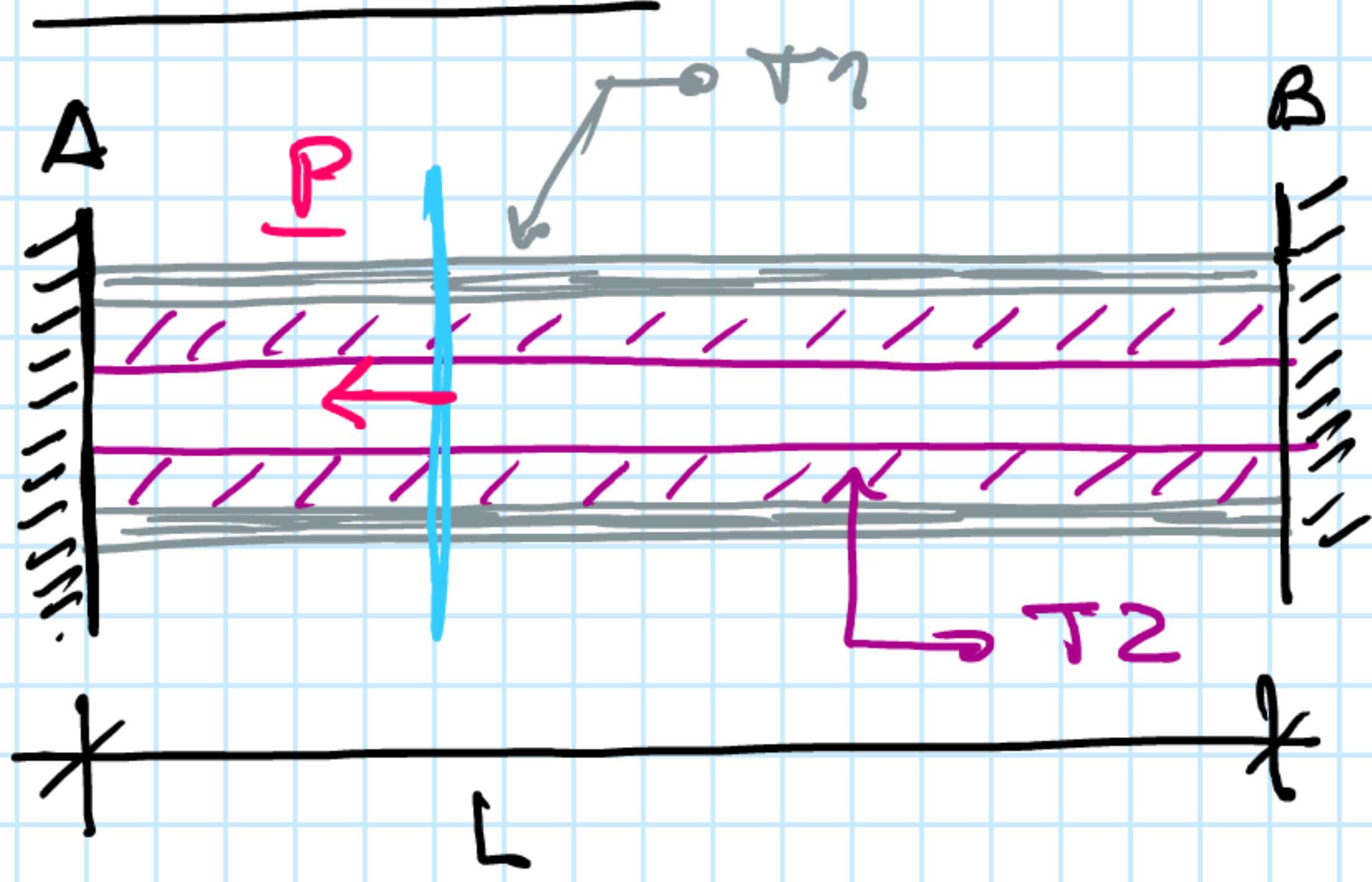
$$\frac{N_{T1} L}{E_1 A_1} = \frac{N_{T2} L}{E_2 A_2}$$

$$\boxed{N_{T1} = \frac{N_{T2}}{E_2 A_2} \cdot E_1 A_1}$$

01.12 -

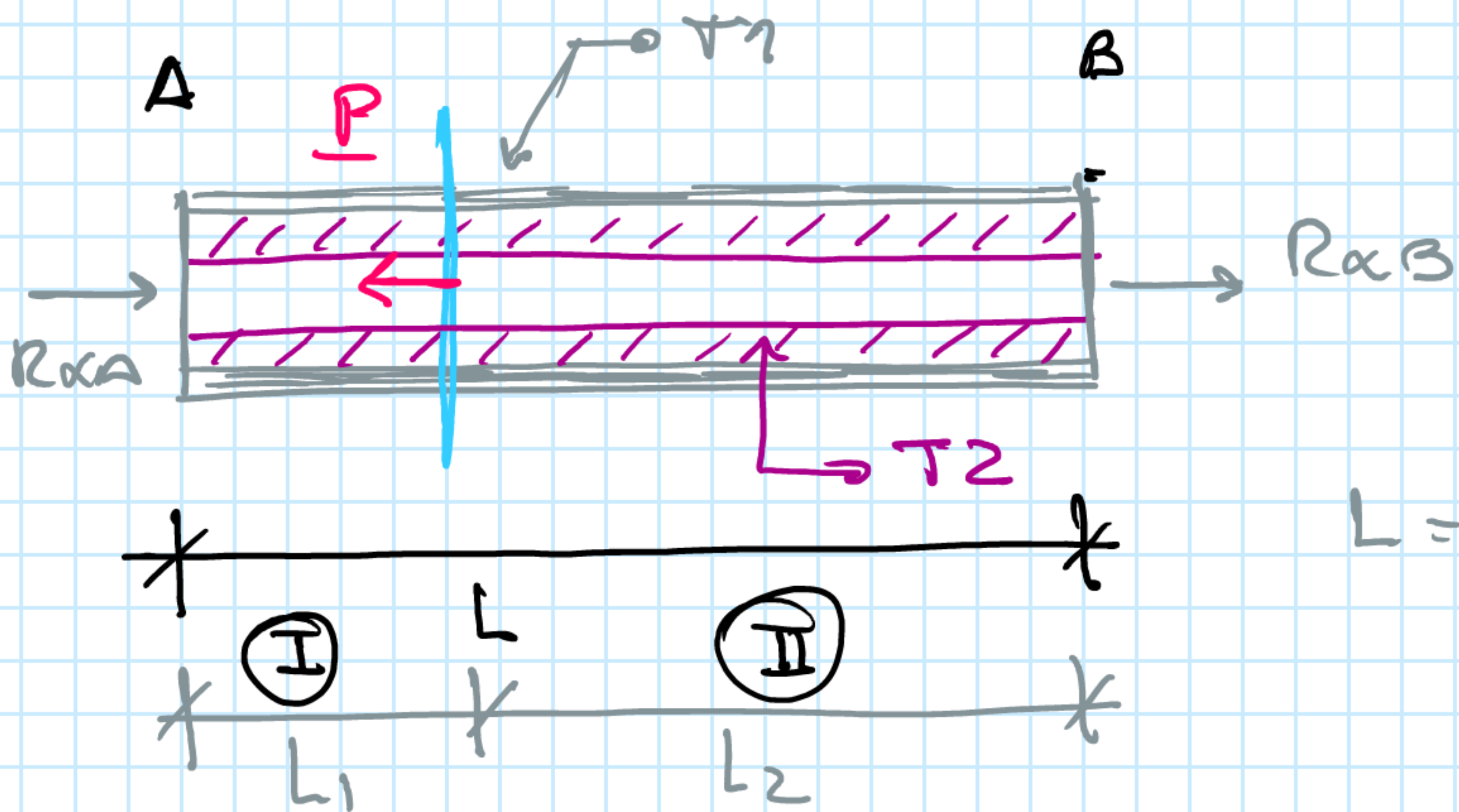
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EC. N° 05:



Hipótesis de deformación por vínculos externos e internos.

$$\sum F_x = 0 \quad R_{xA} - P + R_{xB} = 0.$$



$$L = L_1 + L_2.$$

$$L_{f1} + L_{f2} = L = L_{o1} + L_{o2}$$

$$(L_{f1} - L_{o1}) + (L_{f2} - L_{o2}) = 0.$$

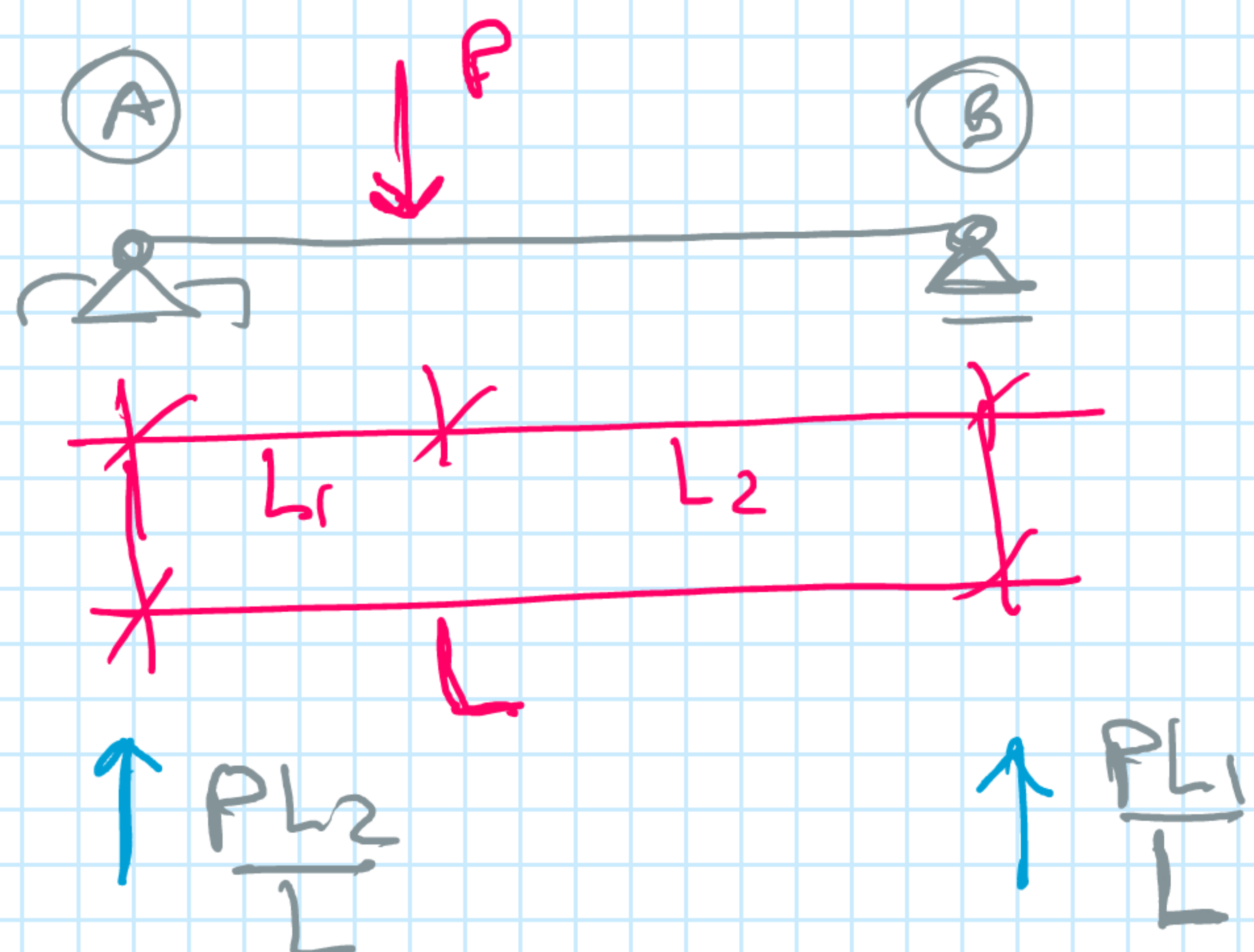
$$\Delta L_1 + \Delta L_2 = 0.$$

$$\Delta L_1 = -\frac{R_{xA} L_1}{EA} \quad ; \quad \Delta L_2 = -\frac{(R_{xA} - P) L_2}{EA}$$

$$-\frac{R_{xA} L_1}{EA} - \frac{(R_{xA} - P) L_2}{EA} = 0.$$

$$-R_{xA} (L_1 + L_2) + PL_2 = 0 \rightarrow R_{xA} = \frac{PL_2}{L_1 + L_2} = \frac{PL_2}{L}$$

$$R_{xA} - P + R_{xB} = 0 \rightarrow R_{xB} = \frac{PL_1}{L_1 + L_2} = \frac{PL_1}{L}$$



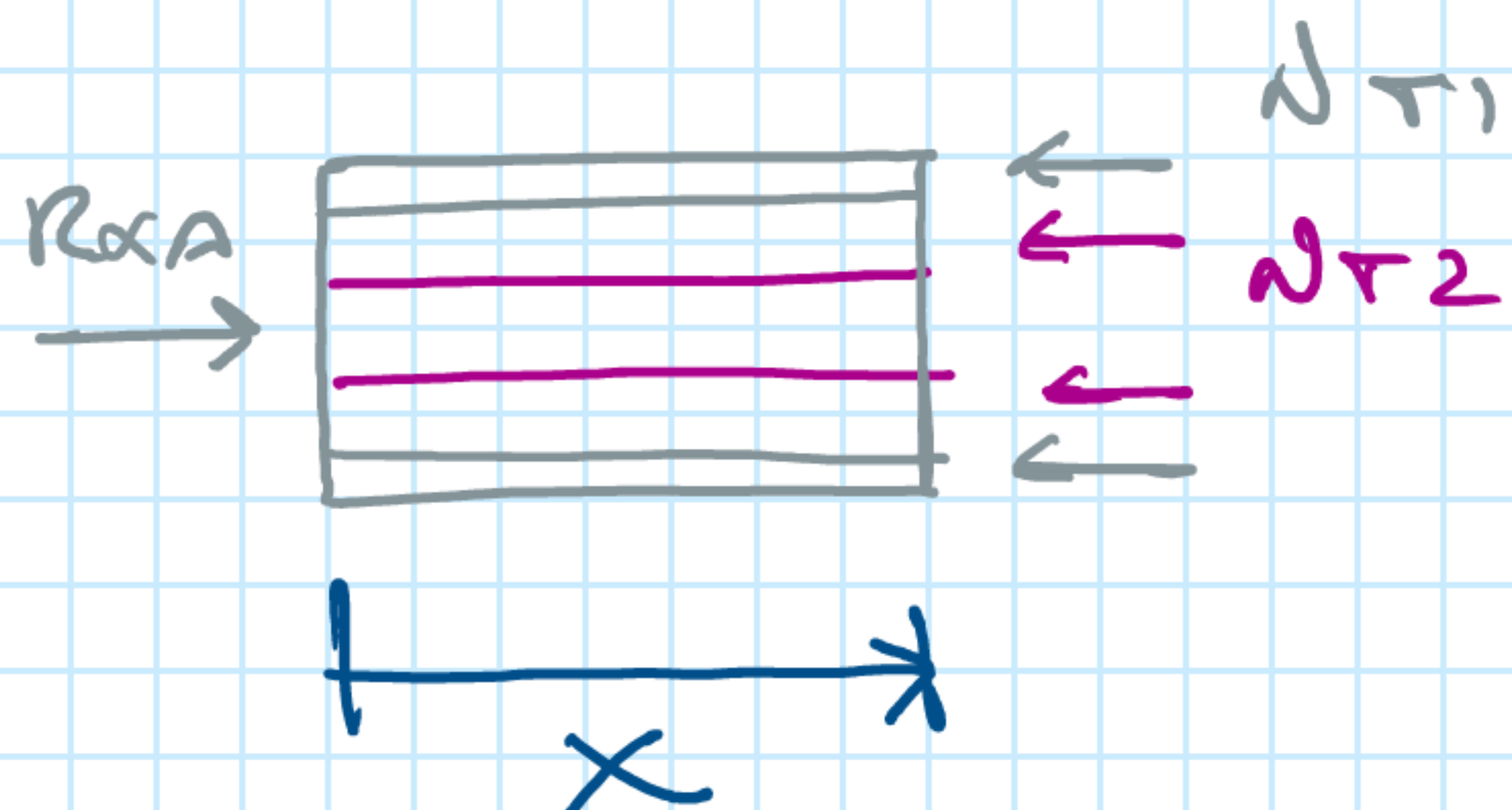
Tramo ①

EC. de Equilibrio:

$$R_{xA} - N_{T1} - N_{T2} = 0.$$

EC. de Compatibilidad:

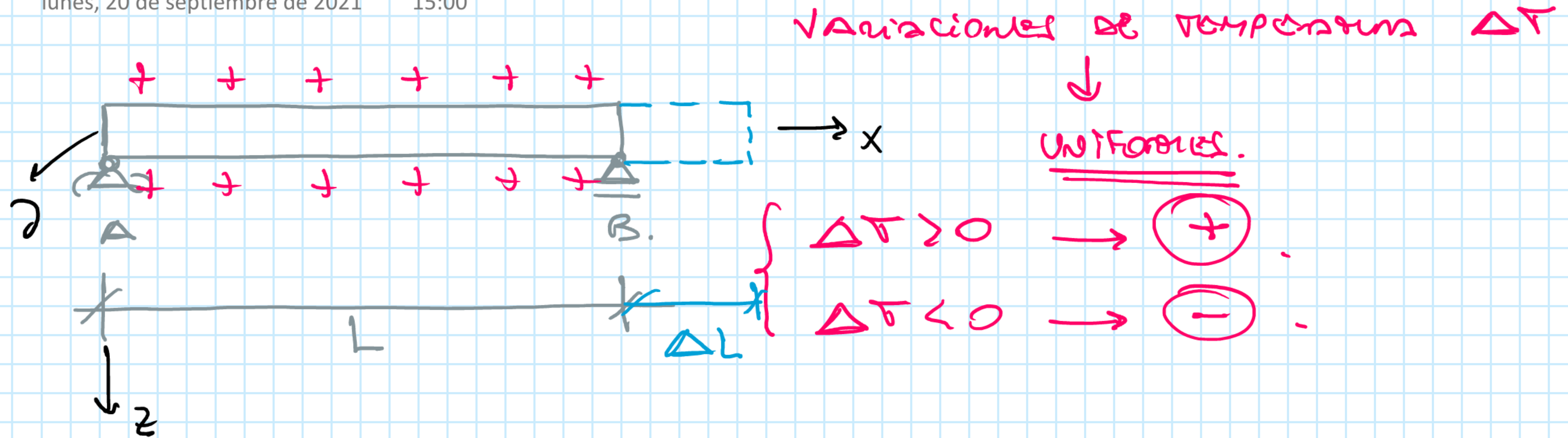
$$\Delta L_{T1} = \Delta L_{T2}$$



$$\frac{N_{T1,1} \cdot x}{E_1 A_1} = \frac{N_{T2,1} \cdot x}{E_2 A_2} \rightarrow N_{T1,1} = N_{T2,1} \frac{E_1 A_1}{E_2 A_2}$$

01.13 - VARIACIÓN DE TEMPERATURA:

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$$\Delta L_x = \alpha \cdot \Delta T \cdot L_0$$

L_0 : Long. inicial de la barra.
 ΔT : a la que se somete la barra
 α : coeficiente de dilatación térmica
 $[\alpha] = \frac{1}{^\circ C}$

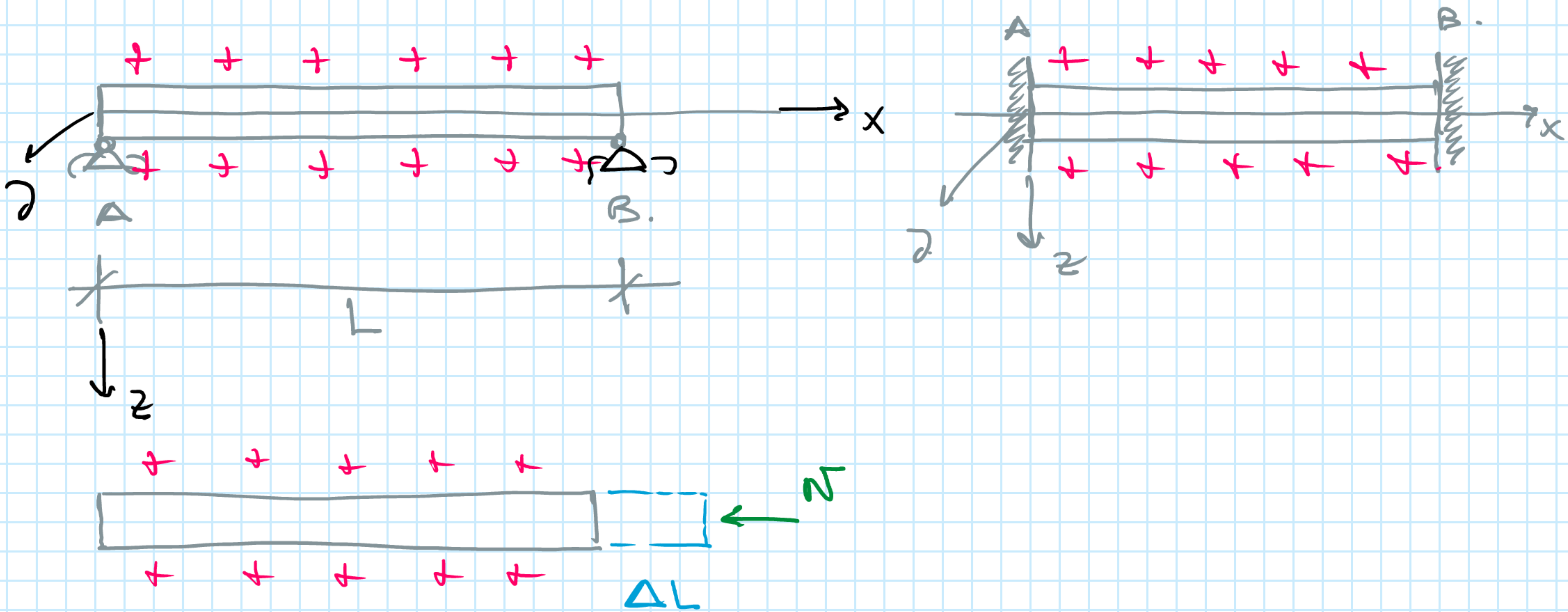
Sistemas isostáticos $\rightarrow$ libremente dilatables.

$$\epsilon_{xx} = \frac{\Delta L}{L_0} = \alpha \cdot \Delta T$$

$$\begin{aligned} \Delta L_y &= \alpha \cdot \Delta T \cdot L_0 & \rightarrow & \left\{ \begin{aligned} \epsilon_{yy} &= \alpha \cdot \Delta T \\ \epsilon_{zz} &= \alpha \cdot \Delta T \end{aligned} \right. \\ \Delta L_z &= \alpha \cdot \Delta T \cdot L_0 \end{aligned}$$

01.13 - VARIACIÓN DE TEMPERATURA:

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$\Delta T \rightarrow \Delta L_{\Delta T} = \alpha \cdot \Delta T \cdot L_0$

$N \rightarrow \Delta L_N = \frac{N \cdot L_0}{EA}$

$\Delta L_{\Delta T} + \Delta L_N = 0.$
 $\alpha \Delta T L_0 + \frac{N L_0}{EA} = 0.$

$N = - \alpha \Delta T L_0 \cdot \frac{EA}{L_0} \rightarrow \boxed{N = - \alpha \Delta T \cdot EA}$

$E > 0 ; A > 0 ; \alpha > 0.$

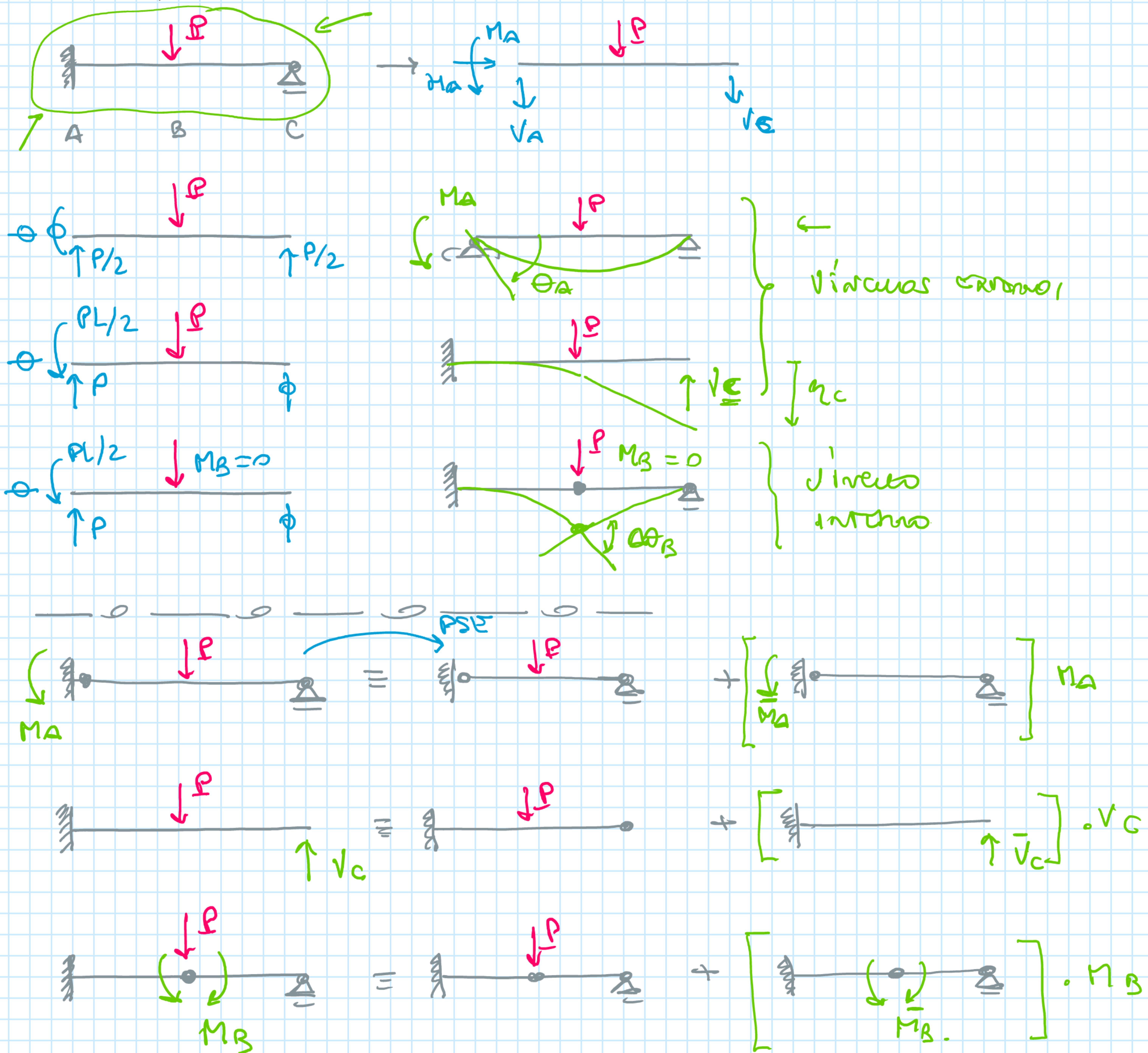
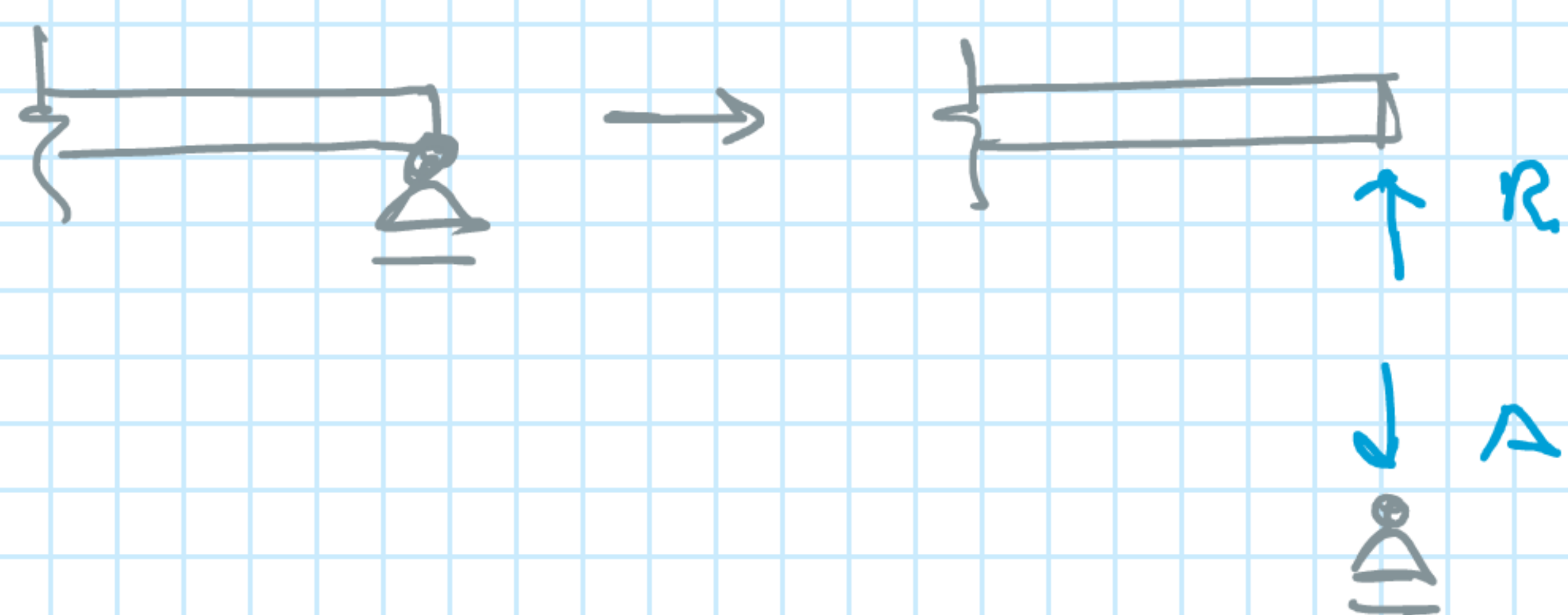
si $\Delta T > 0 \rightarrow N < 0$ Compresión.

si $\Delta T < 0 \rightarrow N > 0$ Tracción.

ΔT	isostático (LIBRO. EJEMPLOS)	hiperestático
RUE	$= 0$	$\neq 0$
$S \equiv RUE$	$= 0.$	$\neq 0$
Tracciones	$= 0.$	$\neq 0.$
$\delta(x)$	$\neq 0$	$\neq 0$ }
e_x	$\neq 0.$	$\neq 0$ }

01.14 - MÉTODO DE LAS INCÓGNITAS ESTÁTICAS - MIE:

lunes, 20 de septiembre de 2021	15:15
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$$P_{R,V} + P_{A,R}.$$


ETS DE COMPARACIÓN DE LOS DESPESAMICADOS:

$$\left\{ \begin{array}{l} \theta_A^H = \theta_{A,\omega}^0 + \theta_{A,\bar{\eta}_A}^0 \cdot \eta_A = 0 \\ \eta_C^H = \eta_{C,\omega}^0 + \eta_{C,\bar{v}_C}^0 \cdot v_C \approx 0 \\ \Delta \theta_B^H = \Delta \theta_{B,\omega}^0 + \Delta \theta_{B,\bar{\eta}_B}^0 \cdot \eta_B = 0 \end{array} \right.$$

Theorem 20 is trivial.

↓
 Answer a 'SA' → have
 to use a conclusion.