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SOLICITACIÓN POR FLEXIÓN EN REGIÓN ANELÁSTICA:

I) → FLEXIÓN SIMPLE.

II) → FLEXIÓN COMPUESTA.

1) PARTIENDO DE LA HBN:

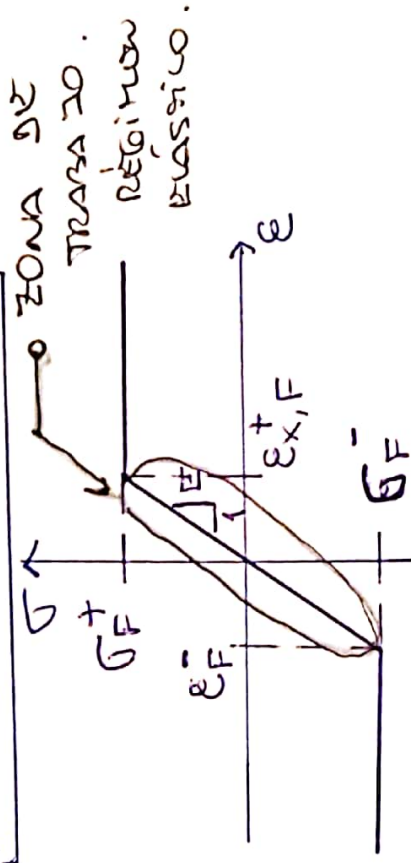
$$\left\{ \begin{aligned} \epsilon_{x,z} &= \frac{z}{z_{\max}} \cdot \epsilon_{x,\max} \\ \epsilon_{x,z} &= \frac{z}{1} \cdot \bar{\epsilon}_x \end{aligned} \right.$$

2) ECS DE EQUIVALENCIA:

$$\left\{ \begin{aligned} N &= 0 = \int_A \sigma_x \cdot dA \\ M_y &= \int_A \sigma_x \cdot z \cdot dA \end{aligned} \right.$$

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3) PERÍODO ELÁSTICO



4) EXPRESIÓN DE FLEXIÓN:

$$\sigma_x = \frac{M_y(x)}{I_y} \cdot z$$

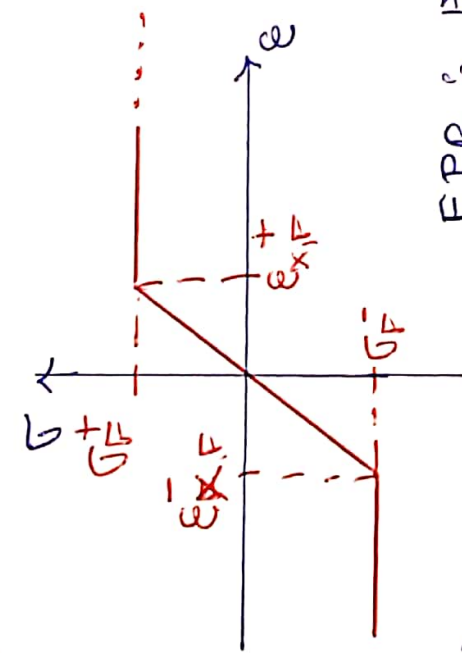
→ FLEXIÓN SIMPLE PURA.

FLEXIÓN SIMPLE ELÁSTICA - PLÁSTICA

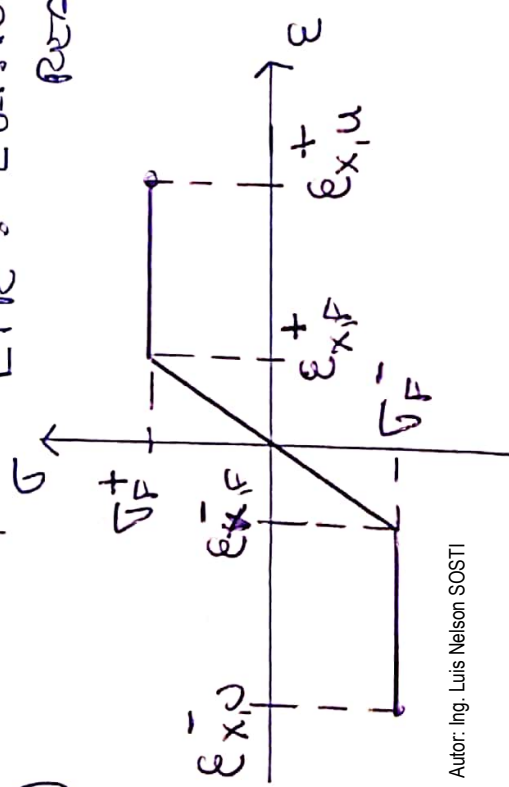
I → TIPOS DE COMPORTAMIENTO

DEL MATERIAL:

1) → EPI: ELASTO-PLÁSTICO IDEAL.



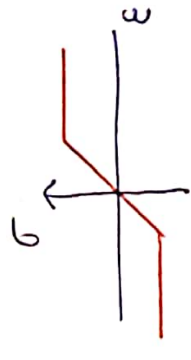
2) EPR: ELASTO-PLÁSTICO REAL.



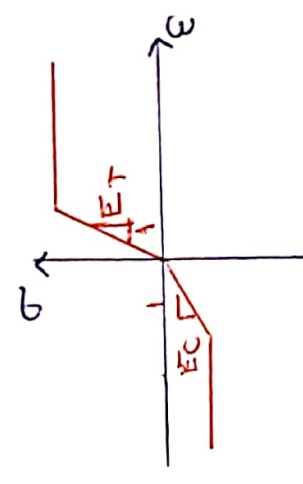
Autor: Ing. Luis Nelson SOSTI

3) - VARIANTES:

a) → $E_T = E_C$.



b) → $E_T \neq E_C$.



c) → $\sigma_F^+ = \sigma_F^-$
 $\sigma_F^+ \neq \sigma_F^-$

d) → $\epsilon_F^+ = \epsilon_F^-$
 $\epsilon_F^+ \neq \epsilon_F^-$

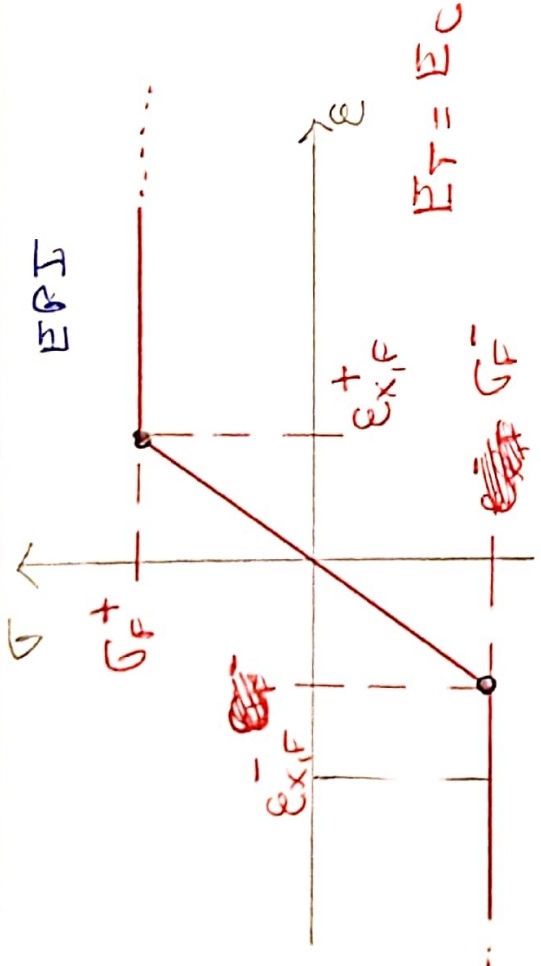
II → TIPOS DE SECCIONES:

1) → DE DOME SIMETRICA.

2) → " SIMPRE SIMETRICA.

3) → CUALQUIERA SI ES DE DOME.

SECCION RECTANGULAR - EPI - DIAGRAMA BILINEAL :

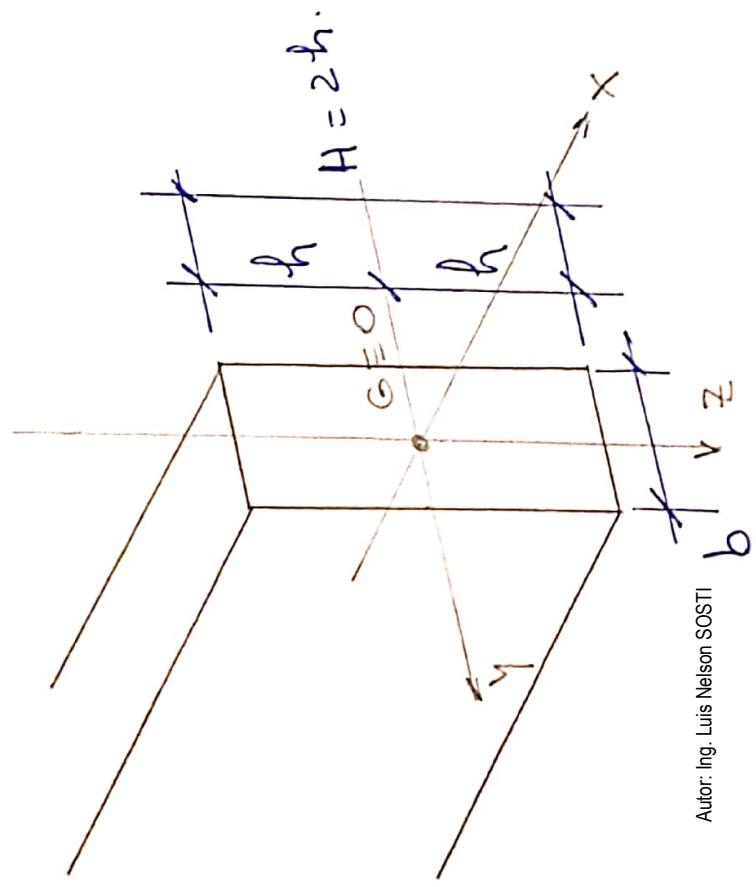


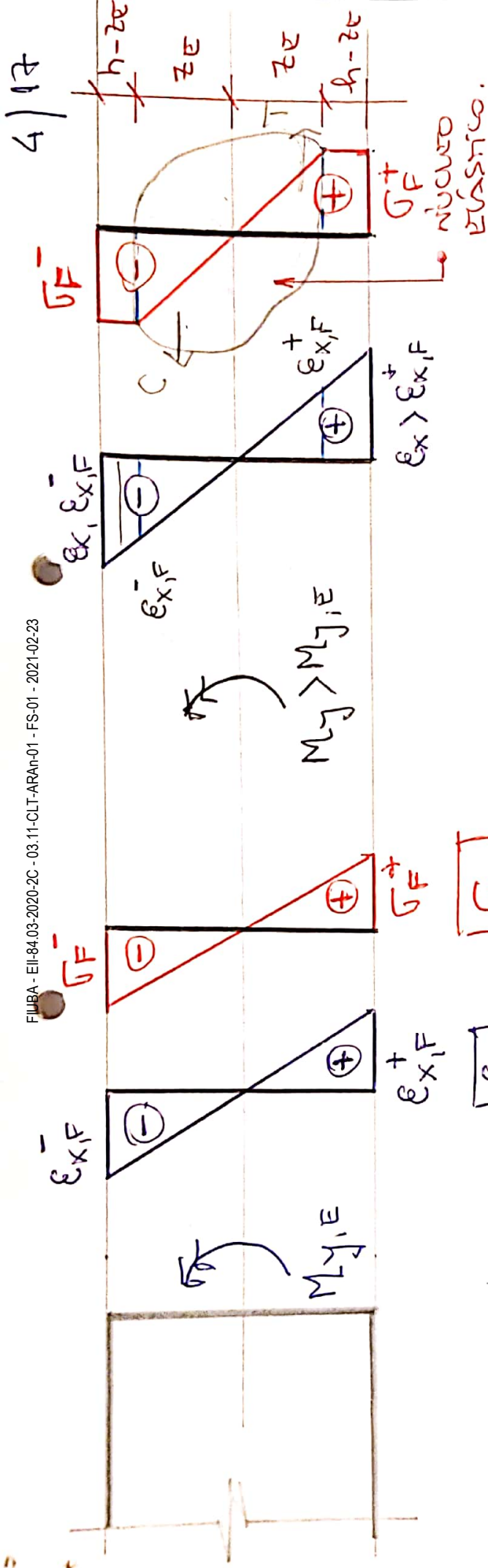
PERIODO ELÁSTICO

- Si $\sigma_{máx} \leq \sigma_F \rightarrow \sigma_{adm} = \frac{M_{y,E}}{I_y} \cdot h$.
- $\sigma_{adm} = \frac{\sigma_F}{CS} \rightarrow \sigma_{adm} = \frac{M_{y,adm}}{I_y} \cdot h$.
- $M_{y,adm} = \frac{I_y}{h} \cdot \sigma_{adm} = \frac{b(2h)^3}{12h} \sigma_{adm}$
- $M_{y,adm} = \frac{2}{3} b h^2 \sigma_{adm}$ (1)
- $\sigma_{máx} = \sigma_F \rightarrow \sigma_F = \frac{M_{y,E}}{I_y} \cdot h$.
- $M_{y,E} = \frac{I_y}{h} \cdot \sigma_F \rightarrow M_{y,E} = \frac{2}{3} b h^2 \sigma_F$ (2)

$M_{y,E}$: MOMENTO FLEXION ELÁSTICO.

$M_{y,E} = M_{y,E}$





• NÚCLEO ELÁSTICO:

$$-z_e \leq z \leq z_e$$

• ZONA PLÁSTICA:

I $z_e \leq z \leq h$

II $-h \leq z \leq -z_e$

• EN REGIÓN ELÁSTICA SIGUE VALIENDO HBW

$$\sigma_{x,F} = \frac{M_{y,E}}{I_y} h = \frac{M_{y,E}}{\frac{I_y}{h}} \cdot \frac{1}{h}$$

S_y : Módulo elástico elástico.

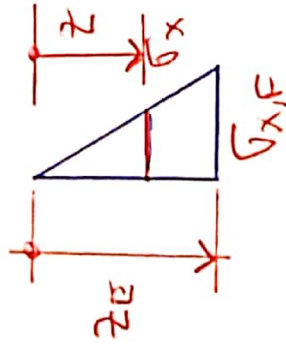
$$\sigma_{x,F} = \frac{M_{y,E}}{S_y} \quad (3)$$

ECS. DE EQUIVALENCIA:

$$M_y = \int_A \sigma_x \cdot z \cdot dA = \int_{-h}^h \sigma_x \cdot z \cdot \frac{b \cdot dz}{dA} = 2 \cdot \int_0^h \sigma_x \cdot z \cdot b \cdot dz$$

$$M_y = 2 \left[\int_0^{z_c} \sigma_x \cdot z \cdot b \cdot dz + \int_{z_c}^h \sigma_x \cdot z \cdot b \cdot dz \right]$$

$$0 \leq z \leq z_c \rightarrow \frac{\sigma_x}{z} = \frac{\sigma_{x,F}}{z_c} \rightarrow \sigma_x = \sigma_{x,F} \cdot \frac{z}{z_c}$$



$$z_c \leq z \leq h \rightarrow \sigma_x = \sigma_{x,F}$$

$$M_y = 2 \left[\int_0^{z_c} \sigma_{x,F} \cdot \frac{z^2}{z_c} \cdot b \cdot dz + \int_{z_c}^h \sigma_{x,F} \cdot z \cdot b \cdot dz \right]$$

$$M_y = 2 \left[\sigma_{x,F} \cdot \frac{b}{z_c} \int_0^{z_c} z^2 \cdot dz + \sigma_{x,F} \cdot b \int_{z_c}^h z \cdot dz \right]$$

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$$M_{y,z} = 2 \left[\frac{\sigma_{x,F} \cdot b}{z_e} \cdot \frac{z^3}{3} \right]_0^{z_e} + \sigma_{x,F} \cdot b \cdot \frac{z^2}{2} \Big|_0^{z_e} = 2 \left[\frac{\sigma_{x,F} \cdot b}{z_e} \cdot \frac{z_e^3}{3} + \sigma_{x,F} \cdot b \left(\frac{z_e^2}{2} - \frac{z_e^2}{2} \right) \right]$$

$$M_y = \frac{2}{3} b \cdot z_e^2 \cdot \sigma_{x,F} + b \cdot (h^2 - z_e^2) \sigma_{x,F}$$

$$M_y = \frac{2}{3} b z_e^2 \cdot \sigma_{x,F} + b h^2 \sigma_{x,F} - b z_e^2 \sigma_{x,F}$$

$$M_y = b h^2 \sigma_{x,F} - \frac{1}{3} b z_e^2 \sigma_{x,F} \quad (4)$$

$$M_y = b h^2 \sigma_{x,F} \left[1 - \frac{1}{3} \frac{z_e^2}{h^2} \right] \quad (5)$$

$$(2) \quad M_{y,z} = \frac{2}{3} b h^2 \sigma_{x,F}$$

$$\frac{M_{y,z}}{M_{y,e}} = \frac{2}{2} \rightarrow M_y = \frac{2}{2} M_{y,e}$$

$$M_y = \frac{2}{2} M_{y,e} \left[1 - \frac{1}{3} \frac{z_e^2}{h^2} \right] \quad (6)$$

Seccions "Parcialment
plastificades".

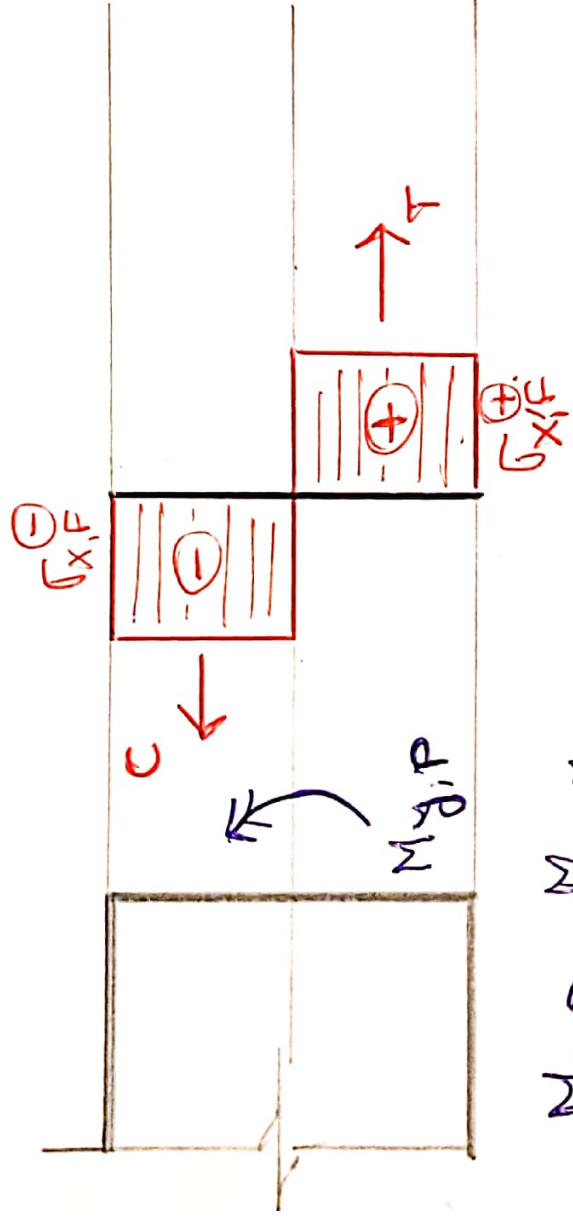
$$z_e = h$$

$$\text{de (6)} \quad M_y = \frac{2}{2} M_{y,e} \left[1 - \frac{1}{3} \right]$$

$$M_y = \frac{2}{2} M_{y,e} \cdot \frac{2}{2}$$

$$M_y = M_{y,e}$$

SECCIONES TOTALMENTE PASTIFICADAS:



$$M_{y,p} = M_{y,u}$$

$$c = T = \sigma_{x,f} \cdot b \cdot h ; \quad z = h ; \quad M_{y,p} = M_{y,u} = c \cdot z = T \cdot z$$

$$M_{y,p} = M_{y,u} = \sigma_{x,f} \cdot b h \cdot h = b h^2 \sigma_{x,f} \quad (7)$$

EN PASTIFICACIÓN POR LA ZC = 0.

$$\text{DE (4)} \quad M_y = b h^2 \sigma_{x,f}$$

$$\text{DE (5)} \quad M_y = b h^2 \sigma_{x,f}$$

$$\text{DE (6)} \quad M_y = \frac{3}{2} M_{y,c}$$

$$M_{y,p} = \frac{3}{2} M_{y,c} \quad (8)$$

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FACTOR DE FORMA:

DE (2) $M_{y,E} = \frac{2}{3} b h^2 \sigma_{x,E}$

DE (7) $M_{y,P} = b h^2 \sigma_{x,P}$

$$\frac{M_P}{M_E} = \frac{b h^2 \sigma_{x,P}}{\frac{2}{3} b h^2 \sigma_{x,E}} = \frac{3}{2} = k$$

FACTOR DE FORMA.

$$\frac{M_P}{\sigma_F} = \frac{\frac{2}{3} b h^2 \sigma_{x,P}}{\sigma_{x,P}} = \frac{2}{3} b h^2 S_y$$

P/ sección rectangular

$$\frac{M_P}{\sigma_F} = \frac{b h^2 \sigma_{x,P}}{\sigma_{x,P}} = b h^2 = Z_y$$

Z_y : Módulo Pásivo de la sección.

$$\begin{aligned} M_P &= Z_y \cdot \sigma_F \\ M_E &= S_y \cdot \sigma_F \end{aligned} \quad (9)$$

P/ sección rectangular

$$\frac{Z_y}{S_y} = \frac{b h^2}{\frac{2}{3} b h^2} = \frac{3}{2} = \frac{M_P}{M_E}$$

P/ sección rectangular $k = 1,50$

P/ " Cuadrada $k = 2,00$

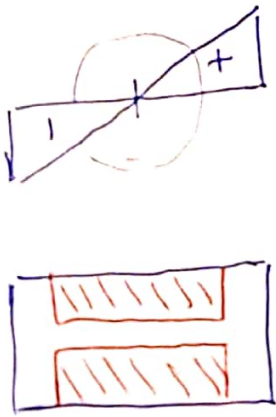
P/ IWA 200 $S_y = 214$
 $Z_y = 250$

$k = 1,168$

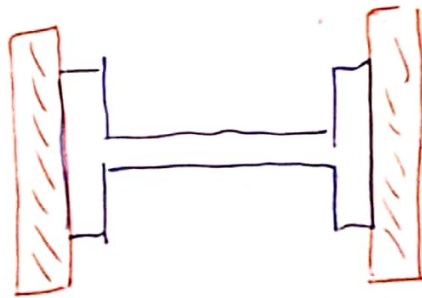
• P/ doble T' $1,08$ a $1,20$.

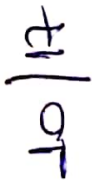
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¿DES ELEGIRÁN UNA SECCIÓN CON UN FACTOR DE FORMA +
GRANDE O MÁS CHICO ???



Se cuanto + chico +
se aprovecha la sección.



$$\frac{d}{dt}$$


$$\frac{d}{dt}$$

$$\frac{d}{dt}$$

$$\frac{d}{dt}$$

$$\frac{d}{dt}$$

$$\frac{d}{dt}$$

$$\frac{d}{dt}$$

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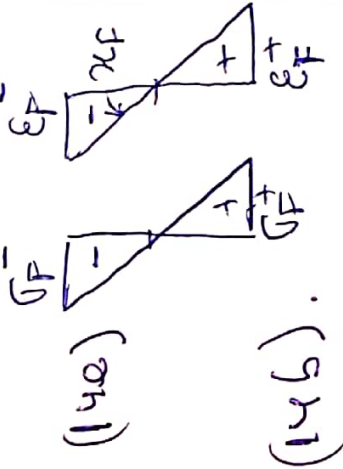
APLIQUEMOS ESTAS EXPRESIONES A NUESTRO PROBLEMA:

• Si $M_y = M_{y,c}$:

$$z_c = h \cdot \epsilon_{x,F} = h \cdot x_F$$

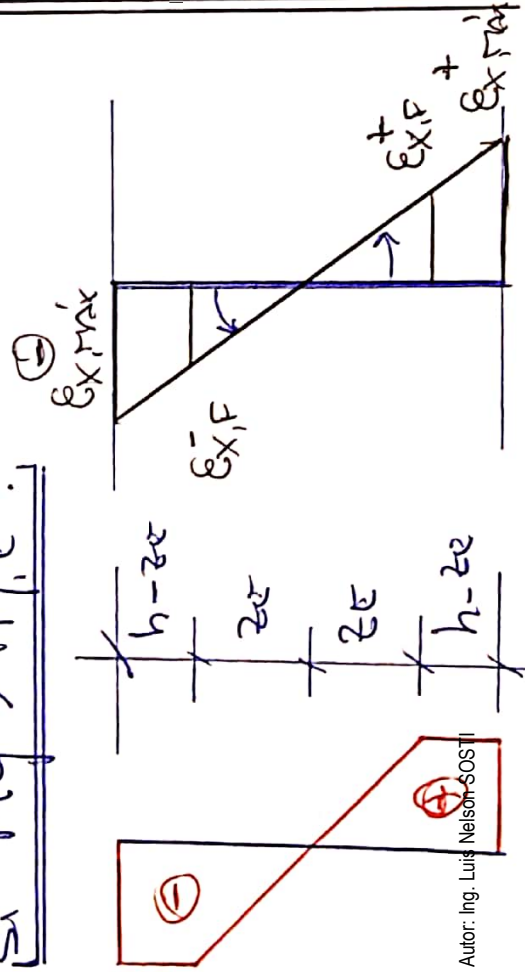
$$x_F = \frac{\epsilon_{x,F}}{h}$$

$$\rho_F = \frac{h}{\epsilon_{x,F}}$$



(14b)

• Si $M_y > M_{y,c}$:



$$\epsilon_{x,F} = z_c \cdot x \rightarrow$$

$$\epsilon_{x,F} = z_c \cdot \frac{1}{\rho} \rightarrow$$

$$\boxed{z_c = \frac{\epsilon_{x,F}}{x}}$$

$$\boxed{z_c = \epsilon_{x,F} \cdot \rho}$$

(15)

$$(14a) \rightarrow h = \frac{\epsilon_{x,F}}{\epsilon_F}$$

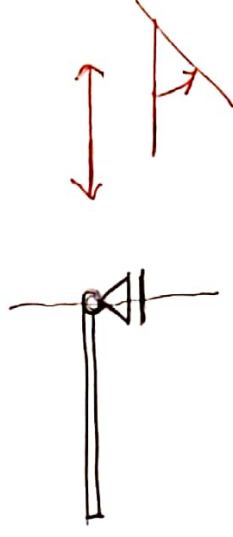
$$(14b) \rightarrow h = \epsilon_{x,F} \cdot \rho_F$$

$$\frac{z_c}{h} = \frac{\epsilon_{x,F}/x}{\epsilon_{x,F}/x_F} = \frac{x_F}{x} \quad (16a)$$

$$\rightarrow \frac{z_c}{h} = \frac{\epsilon_{x,F} \cdot \rho}{\epsilon_{x,F} \rho_F} = \frac{\rho}{\rho_F} \quad (16b)$$

$$\begin{aligned}
 (5) \quad M_y &= b h^2 \sigma_{x,F} \left[1 - \frac{1}{3} \frac{z_e^2}{h^2} \right] = \frac{b h^2 \sigma_{x,F} \left[1 - \frac{1}{3} \frac{x_F^2}{x^2} \right]}{(17a)} \\
 (6) \quad M_y &= \frac{3}{2} M_{y,e} \left[1 - \frac{1}{3} \frac{z_e^2}{h^2} \right] = \frac{\cancel{b h^2 \sigma_x} \frac{3}{2} M_{y,e} \left[1 - \frac{1}{3} \frac{x_F^2}{x^2} \right]}{(17b)}
 \end{aligned}$$

$$\begin{aligned}
 M_y &= b h^2 \sigma_{x,F} \left[1 - \frac{1}{3} \frac{f_F^2}{f^2} \right] \quad (18a) \\
 M_y &= \cancel{\frac{3}{2} M_{y,e}} \frac{3}{2} M_{y,e} \left[1 - \frac{1}{3} \frac{f_F^2}{f^2} \right] \quad (18b)
 \end{aligned}$$



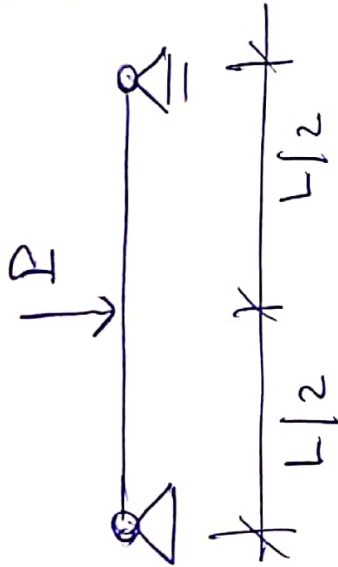
(I) si $M_y = M_{y,e} \rightarrow z_e = h \rightarrow x = x_F \wedge f = f_F.$

(II) si $M_y = M_{y,F} \rightarrow z_e = 0 \quad \frac{z_e}{h} = \frac{x_F}{x} \rightarrow x \rightarrow \infty \wedge$

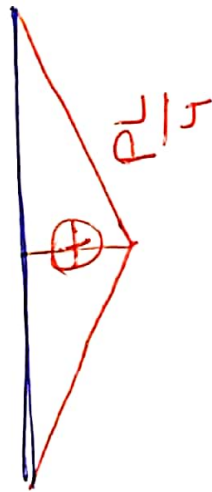
$\frac{z_e}{h} = \frac{f}{f_F} \rightarrow f \rightarrow 0.$

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$GH = 1$



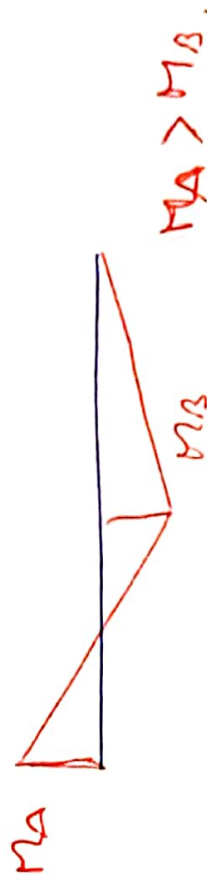
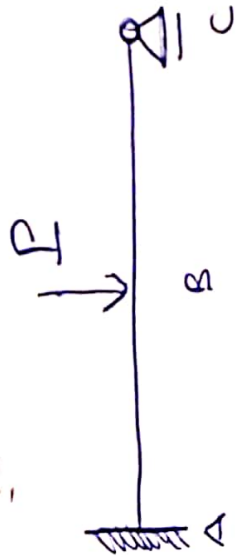
Isostático



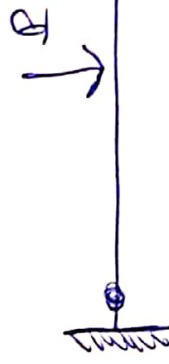
Articulación
plástica



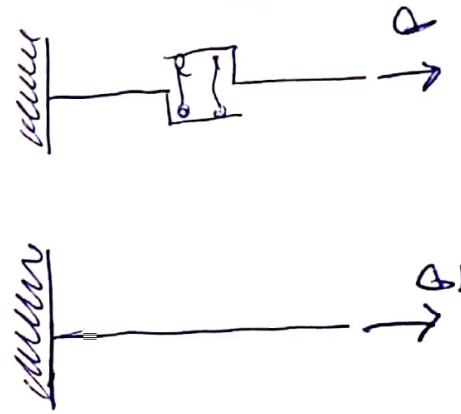
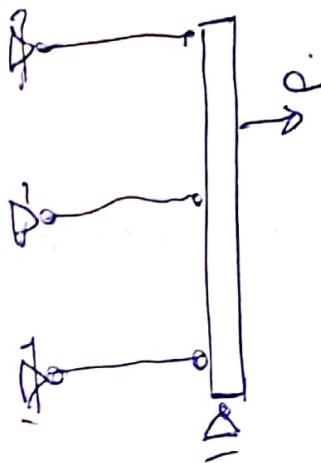
Hiperestático.



Cuando en 'A' se forma una articulación plástica:



$GH = 0$





$$\begin{array}{c} \text{H} \\ \text{H} \end{array} | e$$

$$||$$

$$\Sigma^{\downarrow}$$

$$\uparrow$$

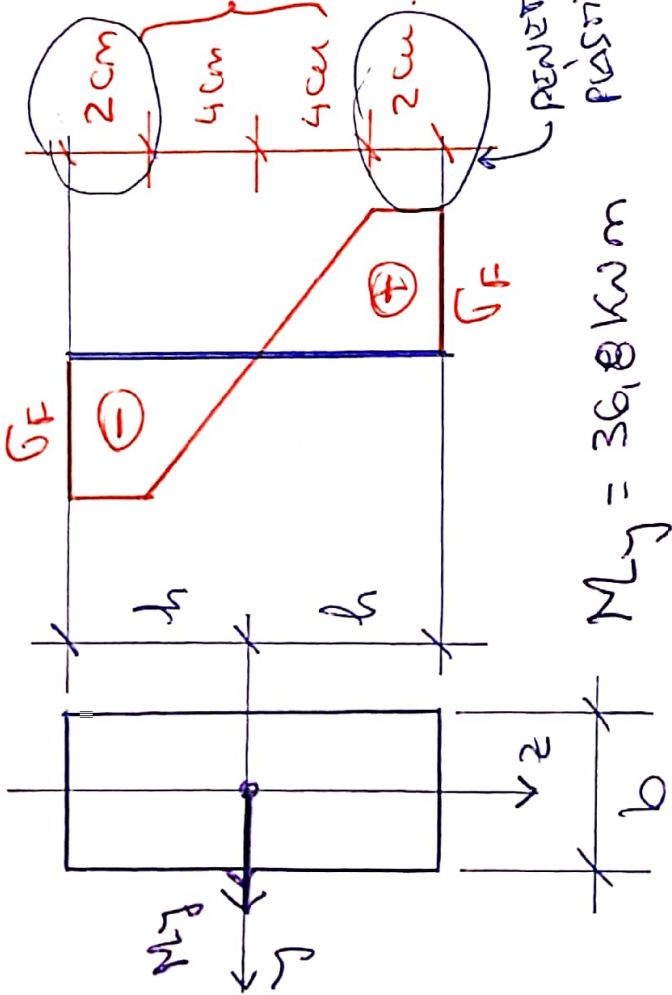
$$\Sigma^{\downarrow} | \text{H} \text{H}$$

$$||$$

$$-|e$$

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EXAMPLE:



DATA:

$$b = 50 \text{ mm}$$

$$h = 60 \text{ mm}$$

$$H = 120 \text{ mm}$$

$$\sigma_F^+ = \sigma_F^- = 24 \text{ kN/cm}^2$$

$$E = 20000 \text{ kN/cm}^2$$

$$M_{y,F} = 36 \text{ kNm}$$

$$I_y = \frac{b(2h)^3}{12} = \frac{2}{3}bh^3 = \frac{2}{3}bh^3$$

$$M_{y,E} = \frac{I_y \cdot \sigma_F}{z_{max}} = \frac{2}{3}bh^3 \sigma_F$$

$$M_{y,E} = \frac{2}{3}bh^2 \sigma_F$$

$$M_{y,E} = \frac{2}{3} \cdot 5 \cdot 6^2 \cdot 24 = 28,80 \text{ kNm}$$

$$M_{y,P} = \frac{3}{2} M_E = 43,20 \text{ kNm}$$

$$M_E \leq M_y \leq M_P$$

$$M_y = \frac{3}{2} M_E \left[1 - \frac{1}{3} \frac{z_F^2}{h^2} \right]$$

$$\frac{1}{3} \frac{z_F^2}{h^2} = 1 - \frac{2}{3} \frac{M_y}{M_E}$$

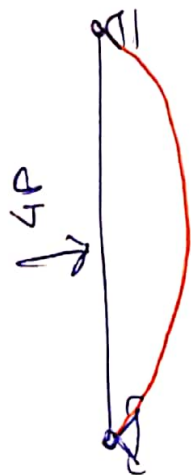
$$z_F = \sqrt{\left(1 - \frac{2}{3} \frac{M_y}{M_E} \right) 3h^2}$$

$$z_F = 40 \text{ mm}$$

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CURVATURA - MAGN. DE CURVATURA - DISTRIBUIÇÃO DE DEFORMAÇÕES:

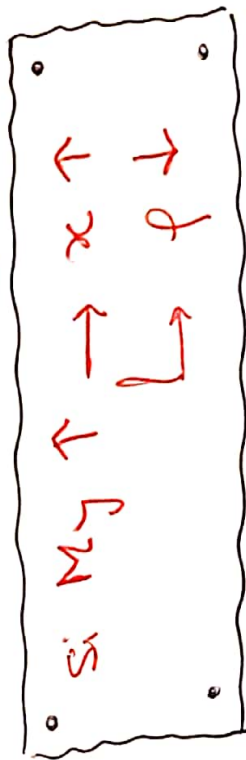
$$\frac{ze}{h} = \frac{x_F}{x} \rightarrow \boxed{x = x_F \cdot \frac{h}{ze}}$$



$$e_{x,F} = \frac{\sigma_F}{E} = \frac{24}{20000} = 1,2 \cdot 10^{-3}$$

$$x_F = \frac{e_{x,F}}{h}$$

$$\boxed{x_F = \frac{1,2 \cdot 10^{-3}}{6cm} = 2 \cdot 10^{-4} \frac{1}{cm}}$$



$$x_F = \frac{M_e}{E I_y} = \text{verificar}$$

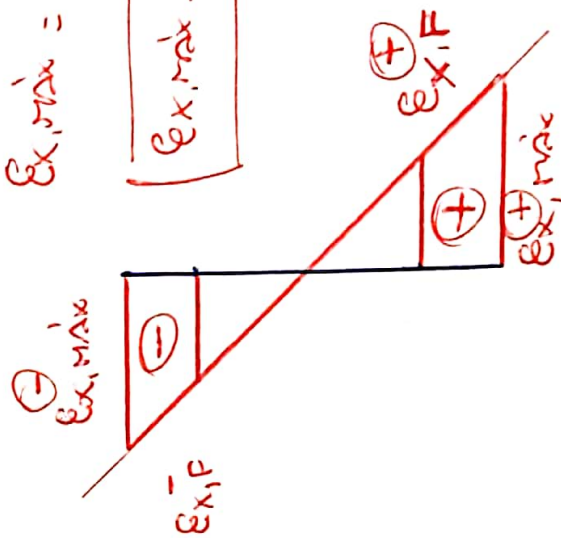
$$x = 2 \cdot 10^{-4} \frac{1}{cm} \cdot \frac{6cm}{4cm} = 3 \cdot 10^{-4} \frac{1}{cm}$$

$$\rho = \frac{1}{x} = 3333 \frac{1}{cm}$$

$$\rho = \frac{1}{x} = 5000 \frac{1}{cm}$$

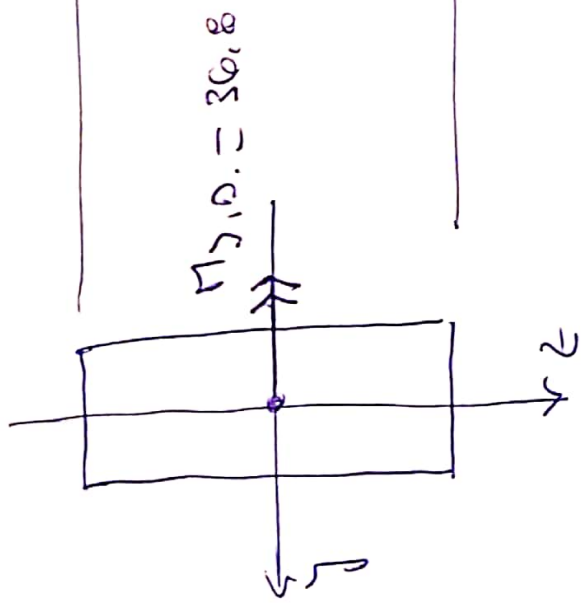
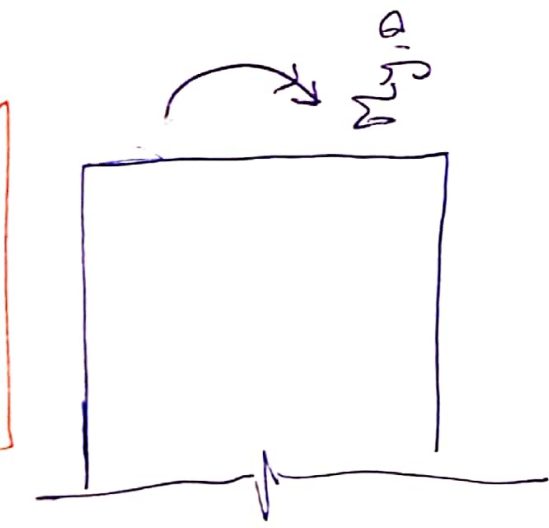
$$e_{x,max} = e_{x,F} \cdot \frac{h}{ze}$$

$$\boxed{e_{x,max} = 1,8 \cdot 10^{-3}}$$



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DESARROLLO



$$M_{y,0} = 36,8$$

